## < SRI KRISHNA INSTITUTE OF TECHNOLOGY, BENGALURU>



## COURSE PLAN

Academic Year 2019-20

| Program: | B E-Mechanical Engineering |
| :---: | :--- |
| Semester : | 3 |
| Course Code: | 18 ME32 |
| Course Title: | Mechanics of Materials |
| Credit / L-T-P: | $4 / 3-2-0$ |
| Total Contact Hours: | 50 |
| Course Plan Author: | CHANDRAIAH M T |

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18ME32: Mechanics of Materials

## A. COURSE INFORMATION

## 1. Course Overview

| Degree: | BE | Program: | ME |
| :--- | :--- | :--- | :--- |
| Semester: | 3 | Academic Year: | $2019-20$ |
| Course Title: | Mechanics of Materials | Course Code: | 18ME32 |
| Credit / L-T-P: | $4 / 3-2-0$ | SEE Duration: | 180 Minutes |
| Total Contact Hours: | 50 Hours | SEE Marks: | 60 Marks |
| CIA Marks: | 40 Marks | Assignment | $1 /$ Module |
| Course Plan Author: | Chandraiah M T | Sign .. | Dt: |
| Checked By: |  | Sign .. | Dt: |
| CO Targets | CIA Target $: 80 \%$ | SEE Target: | $65 \%$ |

Note: Define CIA and SEE \% targets based on previous performance.

## 2. Course Content

Content / Syllabus of the course as prescribed by University or designed by institute. Identify 2 concepts per module as in G.

| $\begin{gathered} \text { Mod } \\ \text { ule } \end{gathered}$ | Content | Teachin g Hours | Identified Module Concepts | Blooms <br> Learning <br> Levels |
| :---: | :---: | :---: | :---: | :---: |
| 1 | Stress, strain, Hook's Law, Poisson's ratio and their relations mechanical properties of engineering materials including elastic constants and their relations, thermal stress and strains and their relation <br> Determine stresses, strains and deformations in bars with varying circular and rectangular cross-sections subjected to normal and temperature loads | $\begin{gathered} 10 \\ (4,6) \end{gathered}$ | -Uni axial stresses \& strains <br> - Deformation | UnderstandL2 <br> Apply L3, |
| 2 | Plane stress and strain, major and minor principle stresses and their planes, maximum shear stress and their planes, Hoop's stress, Longitudinal stress, Radial stress. <br> Determine principal stress, maximum shear stress and their orientations using analytical method and Mohr' circle method and dimensions of thick and thin cylinder and also stress distribution in thick and thin cylinders. | $\begin{gathered} 10 \\ (5,5) \end{gathered}$ | -Biaxial stresses  <br> Hoop's $\&$ Radial <br> Stresses   | $\begin{gathered} \text { L3 } \\ \text { Apply, } \\ \text { L3 } \\ \text { Apply, } \end{gathered}$ |
|  | Circular solid and hallow shafts, Torsional moment of resistance, Power transmission of straight and stepped shafts Maximum Principal Stress theory, Maximum shear stress theory | $\begin{gathered} 10 \\ (5,5) \end{gathered}$ | - Torque \& Twist Theories of failure | $\begin{gathered} \text { L3 } \\ \text { Apply, } \\ \text { L3 } \\ \text { Apply, } \end{gathered}$ |
| 5 | Buckling and stability, Critical load, Columns with pinned ends, Columns with other support conditions and also dimensions of elastic stability of columns using Rankin's and Euler's theory <br> Castigliano's theorem I and II, Load deformation diagram, Strain energy due to normal stresses, Shear stresses, Modulus of resilience, Strain energy due to bending and torsion | $\begin{gathered} 10 \\ (5,5) \end{gathered}$ | -Crippling load -Strain energy | $\begin{gathered} \text { L3 } \\ \text { Apply, } \\ \text { L3 } \\ \text { Apply, } \end{gathered}$ |
| 3 | Draw SFD and BMD for different beams including cantilever beams, simply supported beams and overhanging beams subjected to UDL, UVL, Point loads and couples. And also bending stress, shear stress and its distribution in beams of circular, rectangular, symmetrical I and T sections subjected to point loads and UDL <br> Types of beams, loads, supports, relation between load, force and moment, Pure bending, Curvature of a beam, Longitudinal strains in beams, Normal stresses in Beams with rectangular, circular, ' I ' and ' $T$ ' cross sections | $\begin{gathered} 10 \\ (5,5) \end{gathered}$ | - Shear force \& Bending Moment - Bending stress distribution | L3 Apply, L3 Apply, |
| - | Total | 50 | - | - |

## 3. Course Material

Books \& other material as recommended by university ( $\mathrm{A}, \mathrm{B}$ ) and additional resources used by course teacher (C).

1. Understanding: Concept simulation / video ; one per concept ; to understand the concepts ; 15-30 minutes
2. Design: Simulation and design tools used - software tools used ; Free / open source

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3. Research: Recent developments on the concepts - publications in journals; conferences etc.

| Module <br> s | Details | Chapters in book | Availability |
| :---: | :---: | :---: | :---: |
| A | Text books (Title, Authors, Edition, Publisher, Year.) | - | - |
| $\begin{gathered} 1,2,3,4 \\ 5 \end{gathered}$ | Strength of Material, James M Gere, Barry J Goodno, Indian Edition, Cengage Learning, 2009 | $\begin{gathered} 1,23 \\ 5,7,9,11 \\ 13,14 \end{gathered}$ | In Lib / In Dept |
| $\begin{gathered} 1,2,3,4, \\ 5 \end{gathered}$ | Strength of Materials, R Subramanian, Oxford. 2005 | $\begin{gathered} 1,2,3 \\ 4, \\ 5,8,9,11 \end{gathered}$ | $\begin{aligned} & \text { In Lib/ In } \\ & \text { Dept } \end{aligned}$ |
|  |  |  |  |
| B | Reference books (Title, Authors, Edition, Publisher, Year.) | - | - |
| $\frac{1,2,3,4,}{5}$ | Strength of Materials,S S Rattan, Second Edition, McGraw Hill, 2011. | $\begin{gathered} 1,2,4, \\ 5,6,8,11 \\ 12, \end{gathered}$ | In Lib |
| $\underset{5}{1,2,3,4,}$ | Mechanics of Materials, Ferdinand Beer and russell Johnston, Tata McGraw Hill. 2003 | $\begin{gathered} 1,2,3, \\ 5,7,9,10 \\ 12 \end{gathered}$ | In Lib |
|  |  |  |  |
| C | Concept Videos or Simulation for Understanding | - | - |
| C1 | https://www.youtube.com/watch?v=cMdVzMRWZTk -17.23 Mins |  |  |
| C2 | https://www.youtube.com/watch?v=SBiU3M7D16s -3.14 Mins |  |  |
| C3 | https://www.youtube.com/watch?v=C207JS-HM4Q -8.27 Mins |  |  |
| C4 | https://www.youtube.com/watch?v=Ja03J1RQ3Hw -8.38 Mins |  |  |
| C5 | https://www.youtube.com/watch?v=ICDZ5uLGrI4 -5.26 Mins |  |  |
| C6 | https://www.youtube.com/watch?v=qbv2rOEMyiA -13.36 Mins |  |  |
| C7 | https://www.youtube.com/watch?v=Fzjd-6hzVko -7.27 Mins |  |  |
| C8 | https://www.youtube.com/watch?v=WB__FR_1_LU -22.22 Mins |  |  |
| C9 | https://www.youtube.com/watch?v=UahfUvcS24o -16.42 Mins |  |  |
| C10 | https://www.youtube.com/watch?v=xGPGrS-k6eo -10.31 Mins |  |  |
| D | Software Tools for Design | - | - |
| 1 | https://www.youtube.com/watch?v=bMZaiTsMo-U -3.58 Mins |  |  |
|  |  |  |  |
| E | Recent Developments for Research | - | - |
| 1 | https://www.youtube.com/watch?v=J0ZMi83oUjk -4.38 Mins |  |  |
|  |  |  |  |
| F | Others (Web, Video, Simulation, Notes etc.) | - | - |
| 1 | https://www.youtube.com/watch? v=I11NU1Kj8P8 -10.38 Mins |  |  |
|  |  |  |  |

## 4. Course Prerequisites

Refer to GL01. If prerequisites are not taught earlier, GAP in curriculum needs to be addressed. Include in Remarks and implement in B.5.
Students must have learnt the following Courses / Topics with described Content . .

| Modu <br> les | Course <br> Code | Course Name | Topic / Description | Sem | Remarks | Blooms <br> Level |
| :---: | :---: | :--- | :--- | :--- | :--- | :---: |
| 1 | 18 CIV13/ <br> 23 | Elements of Civil <br> Engineering |  | $1 / 2$ |  | L3 Knowledge force, Moments |
|  |  |  |  |  |  |  |

## 5. Content for Placement, Profession, HE and GATE

The content is not included in this course, but required to meet industry \& profession requirements and help students for Placement, GATE, Higher Education, Entrepreneurship, etc. Identifying Area / Content requires experts consultation in the area.
Topics included are like, a. Advanced Topics, b. Recent Developments, c. Certificate Courses, d. Course Projects, e. New Software Tools, f. GATE Topics, g. NPTEL Videos, h. Swayam videos etc.

| Modu <br> les | Topic / Description |  | Area | Remarks | Blooms <br> Level |
| :---: | :---: | :---: | :--- | :---: | :---: |
| 1 | Advanced strength of <br> nptel.ac.in/courses/112101095/ | materials | Higher Study, <br> Gate | Gap | Apply <br> L3 |
| - |  |  |  |  |  |

## B. OBE PARAMETERS

## 1. Course Outcomes

Expected learning outcomes of the course, which will be mapped to POs. Identify a max of 2 Concepts per Module.
Write 1 CO per Concept.

| $\begin{gathered} \text { Modu } \\ \text { les } \end{gathered}$ | Course Code.\# | Course Outcome <br> At the end of the course, student should be able to ... | Teach. Hours | Concept | Instr Method | $\begin{gathered} \text { Assessment } \\ \text { Method } \end{gathered}$ | Blooms' Level |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 18ME32.1 | Understand simple, thermal stresses \& strains and their relations | 04 | Unidirection al Stresses \& Strain | Chalk and <br> Board | Assignmen t, Unit Test \& CIE | L2 Understand |
| 1 | 18ME32.2 | Determine the stress, strain and deformation in bars with varying cross section and temperature change | 06 | Deformation | Chalk and Board | Assignmen t , Unit Test \& CIE | $\begin{gathered} \text { L3 } \\ \text { Apply } \end{gathered}$ |
| 2 | 18ME32.3 | Determine principal stresses and planes using analytical and graphical method. | 05 | Biaxial Stresses | Chalk and <br> Board | Assignmen t , Unit Test \& CIE | $\begin{gathered} \text { L3 } \\ \text { Apply } \end{gathered}$ |
| 2 | 18ME32.4 | Determine stress distribution in thick and thin cylinder | 05 | $\begin{array}{lr} \hline \text { Hoop's } & \& \\ \text { Radial Stress } \end{array}$ | Chalk and <br> Board | Assignmen t, Unit Test \& CIE | $\begin{gathered} \text { L3 } \\ \text { Apply } \end{gathered}$ |
| 3 | 18ME32.5 | Determine, torsional strength, rigidity, flexibility \& dimensions of shaft | 05 | Torque \& Twist | Chalk and <br> Board | Assignmen t, Unit Test \& CIE | $\begin{gathered} \text { L2 } \\ \text { Apply } \end{gathered}$ |
| 3 | 18ME32.6 | Determine dimensions of bars, beams \& rods using Maximum principal \& Maximum Shear stress theory of failure | 05 | Theories of failure | Chalk and <br> Board | $\begin{gathered} \text { Assignmen } \\ \mathrm{t} \text {, Unit Test } \\ \text { \& CIE } \end{gathered}$ | $\begin{gathered} \text { L3 } \\ \text { Apply } \end{gathered}$ |
| 4 | 18ME32.7 | Determine elastic stability of columns using Rankine's \& Euler's theory | 05 | Crippling load | Chalk and <br> Board | Assignmen t , Unit Test \& CIE | $\begin{gathered} \text { L3 } \\ \text { Apply } \end{gathered}$ |
| 4 | 18ME32.8 | Determine dimensions of bars, beams \& rods using strain energy methods | 05 | Strain energy methods | Chalk and <br> Talk | $\begin{gathered} \text { Assignmen } \\ t \& C I E \end{gathered}$ | $\begin{gathered} \text { L3 } \\ \text { Apply } \end{gathered}$ |
| 5 | 18ME32.9 | Draw shear force diagram \& Bending moment diagram for a Simply supported beam \& cantilever beam with point load, UDL \& UVL | 05 | Shear force, \& Bending moment | Chalk and Board | Assignmen t, Unit Test \& CIE | $\begin{gathered} \text { L3 } \\ \text { Apply } \end{gathered}$ |
| 5 | 18ME32.10 | Determine the bending stress distribution of rectangular, symmetrical I \& T section | 05 | Bending stress distribution | Chalk and Board | Assignmen t , Unit Test \& CIE | $\begin{gathered} \text { L3 } \\ \text { Apply } \end{gathered}$ |
| - | - | Total | 50 | - | - | - | L2-L3 |

## 2. Course Applications

Write 1 or 2 applications per CO.
Students should be able to employ / apply the course learning's to . . .

| Modu <br> les | Application Area <br> Compiled from Module Applications. | CO | Level |
| :---: | :---: | :---: | :---: |
| 1 | Stress and strain to find an application in analyzing as structure | CO1 | L2 |

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| 1 | Structural analysis | CO 2 | L 3 |
| :--- | :--- | :---: | :---: |
| 2 | Combined loading in structural and machine parts | CO 3 | L 3 |
| 2 | Pressure vessels | CO 4 | L 3 |
| 4 | Design of shaft | CO 5 | L 3 |
| 4 | Failure theories analysis | CO 6 | L 3 |
| 5 | Design of columns | CO 7 | L 3 |
| 5 | Strain energy stored in the materials | CO 8 | L 3 |
| 3 | Design of Bridges, | CO 9 | L 3 |
| 3 | Structural analysis in the field of construction of bridges, etc. | CO 10 | L 3 |

## 3. Mapping And Justification

CO - PO Mapping with mapping Level along with justification for each CO-PO pair.
To attain competency required (as defined in POs) in a specified area and the knowledge \& ability required to accomplish it.

| Mod ules | Mapping |  | Mapping Level | Justification for each CO-PO pair | Lev el |
| :---: | :---: | :---: | :---: | :---: | :---: |
| - | CO | PO | - | 'Area': 'Competency' and 'Knowledge' for specified 'Accomplishment' |  |
| 1 | CO1 | PO1 | 3 | 'Engineering Knowledge:' - Acquisition of Engineering Knowledge of Stress and strain is essential to accomplish solutions to complex engineering problems in Mechanical Engineering. | L2 |
| 1 | CO1 | PO2 | 2 | Problem Analysis': Analyzing problems require knowledge / understanding of relation between stress and strain to accomplish solutions to complex engineering problems in Mechanical Engineering. | L3 |
| 1 | CO2 | PO1 | 3 | 'Engineering Knowledge:' - Acquisition of Engineering Knowledge of Deformation of materials is essential to accomplish solutions to complex engineering problems in Mechanical Engineering. | L3 |
| 1 | CO2 | PO2 | 2 | Problem Analysis': Analyzing problems require knowledge / understanding of deformation of different section of materials to accomplish solutions to complex engineering problems in Mechanical Engineering. | L3 |
| 2 | CO3 | PO1 | 3 | 'Engineering Knowledge:' - Acquisition of Engineering Knowledge of Compound stresses is essential to accomplish solutions to complex engineering problems in Mechanical Engineering. | L3 |
| 2 | CO3 | PO2 | 2 | Problem Analysis': Analyzing problems require knowledge / understanding of Biaxial loads to accomplish solutions to complex engineering problems in Mechanical Engineering. | L3 |
| 2 | CO4 | PO1 | 3 | 'Engineering Knowledge:' - Acquisition of Engineering Knowledge of Hoop Stress and radial stress is essential to accomplish in stress distribution thin and thick cylinder | L3 |
| 2 | CO4 | PO2 | 2 | Problem Analysis': Analyzing problems require knowledge / understanding of Hoops and radial load to accomplish solutions to complex engineering problems in Mechanical Engineering. | L3 |
| 3 | CO5 | PO1 | 3 | 'Engineering Knowledge:' - Acquisition of Engineering Knowledge of Torque and twist is essential to accomplish dimensions of solid and hollow shafts in Mechanical Engineering. | L3 |
| 3 | CO5 | PO2 | 2 | Problem Analysis': Analyzing problems require knowledge / understanding of torque and twist to accomplish solutions to complex engineering problems in Mechanical Engineering. | L3 |
| 3 | CO6 | PO1 | 3 | 'Engineering Knowledge:' - Acquisition of Engineering Knowledge of Theories of failure is essential to accomplish solutions to complex engineering problems in Mechanical Engineering. | L3 |
| 3 | CO6 | PO2 | 2 | Problem Analysis': Analyzing problems require knowledge / understanding of theories of failure to accomplish solutions to complex engineering problems in Mechanical Engineering. | L3 |
| 4 | CO7 | PO1 | 3 | 'Engineering Knowledge:' - Acquisition of Engineering Knowledge of crippling load is essential to accomplish different end condition of columns. | L3 |
| 4 | CO7 | PO2 | 2 | Problem Analysis': Analyzing problems require knowledge / understanding of Crippling load to accomplish different end condition of columns. | L3 |
| 4 | CO8 | PO1 | 3 | 'Engineering Knowledge:' - Acquisition of Engineering Knowledge of Strain energy is essential to accomplish solutions to find the energy absorption in different load condition | L3 |

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| 4 | CO8 | PO 2 | 2 | Problem Analysis': Analyzing problems require knowledge / understanding of Strain energy to accomplish solutions to find the energy absorption in different load condition | L3 |
| :---: | :---: | :---: | :---: | :---: | :---: |
| 5 | CO9 | PO1 | 3 | 'Engineering Knowledge:' - Acquisition of Engineering Knowledge of SFD and BMD is essential to accomplish solutions to beams with different load condition | L3 |
| 5 | CO9 | PO2 | 2 | Problem Analysis': Analyzing problems require knowledge / understanding of SFD and BMD to accomplish solutions to beams with different load condition | L3 |
| 5 | CO10 | PO1 | 3 | 'Engineering Knowledge:' - Acquisition of Engineering Knowledge of Bending stress is essential to accomplish solutions to complex engineering problems in Mechanical Engineering. | L3 |
| 5 | CO10 | PO2 | 2 | Problem Analysis': Analyzing problems require knowledge / understanding of bending stress distributionto accomplish solutions to complex engineering problems in Mechanical Engineering. | L3 |

## 4. Articulation Matrix

CO - PO Mapping with mapping level for each CO-PO pair, with course average attainment.

| - | - | Course Outcomes | Program Outcomes |  |  |  |  |  |  |  |  |  |  |  |  |  |  | - |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Modu | CO.\# | At the end of the course student |  |  |  |  |  | PO | PO | PO |  |  |  | PO | PS | PS |  | Lev |
| les |  | should be able to . . . | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 | 10 | 11 | 12 | O1 | O2 | O3 | el |
| 1 | 18ME32.1 | Understand simple, thermal stresses \& strains and their relations | 3 |  |  |  |  |  |  |  |  |  |  |  |  |  |  | L2 |
| 1 | 18ME32.2 | Determine the stress, strain and deformation in bars with varying cross section and temperature change | 3 | 2 |  |  |  |  |  |  |  |  |  |  |  |  |  | L3 |
| 2 | 18ME32.3 | Determine principal stresses and planes using analytical and graphical method. | 3 | 2 |  |  |  |  |  |  |  |  |  |  |  |  |  | L3 |
| 2 | 18ME32.4 | Determine stress distribution in thick and thin cylinder | 3 | 2 |  |  |  |  |  |  |  |  |  |  |  |  |  | L3 |
| 3 | 18ME32.5 | Determine, torsional strength, rigidity, flexibility \& dimensions of shaft | 3 | 2 |  |  |  |  |  |  |  |  |  |  |  |  |  | L3 |
| 3 | 18ME32.6 | Determine dimensions of bars, beams \& rods using Maximum principal \& Maximum Shear stress theory of failure | 3 | 2 |  |  |  |  |  |  |  |  |  |  |  |  |  | L3 |
| 4 | 18ME32.7 | Determine elastic stability of columns using Rankine's \& Euler's theory | 3 | 2 |  |  |  |  |  |  |  |  |  |  |  |  |  | L3 |
| 4 | 18ME32.8 | Determine dimensions of bars, beams \& rods using strain energy methods | 3 | 2 |  |  |  |  |  |  |  |  |  |  |  |  |  | L3 |
| 5 | 18ME32.9 | Draw shear force diagram \& Bending moment diagram for a Simply supported beam \& cantilever beam with point load, UDL \& UVL |  | 2 |  |  |  |  |  |  |  |  |  |  |  |  |  | L3 |
| 5 | 18ME32.10 | Determine the bending stress distribution of circular, rectangular symmetrical I \&T section |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  | L3 |
| - | 18ME32 | Average attainment (1, 2, or 3) | 3 | 2 |  |  |  |  |  |  |  |  |  |  |  |  |  | - |
| - | PO, PSO | 1.Engineering Knowledge; 2.Problem Investigations of Complex Problem 7.Environment and Sustainability; 11.Project Management and Finance Base Management; S3.Web Design | $\begin{aligned} & n A n \\ & n s ; \\ & 8 . E t \\ & \text { e; } \end{aligned}$ | naly <br> 5.M <br> hics <br> 2.Li | sis; Mode s; 9 ife-lo | 3.D <br> rn <br> 9.Indi <br> ong |  | $\begin{aligned} & \text { ign / } \\ & \text { ol U } \\ & \text { dual } \\ & \text { arnin } \end{aligned}$ | $D e l$ | velo <br> e; <br> d <br> S1.S | $\begin{aligned} & \text { ppmer } \\ & \text { 6.Th } \\ & \text { Team } \\ & \text { Softw } \end{aligned}$ |  | $\begin{aligned} & \text { of S } \\ & \text { Eng } \\ & \text { rk; } \\ & \text { e Er } \end{aligned}$ |  | on | $4$ | Con <br> Soc <br> nica <br> S2. | nduct <br> ciety; <br> ation, <br> .Data |

## 5. Curricular Gap and Content

Topics \& contents not covered (from A.4), but essential for the course to address POs and PSOs.

| Modu <br> les | Gap Topic | Actions Planned | Schedule Planned | Resources Person | PO Mapping |
| :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | Advanced mechanics | Seminar | $2^{\text {nd }}$ Nov 2019 | Dr. M H Annaiah | PO1 |

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$\left.\begin{array}{|c|c|c|c|c|c|}\hline & & \begin{array}{c}\text { Prof. And Vice } \\ \text { Principal }\end{array} \\ \text { SCE, Bengaluru }\end{array}\right]$.

## 6. Content Beyond Syllabus

Topics \& contents required (from A.5) not addressed, but help students for Placement, GATE, Higher Education, Entrepreneurship, etc

| Modu <br> les | Gap Topic | Area | Actions Planned | Schedule Planned | Resources Person | PO Mapping |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | Advances in <br> Construction of beams <br> and columns | Placement, <br> GATE, <br> Higher <br> Study, <br> Entrepreneur <br> ship. | Presentation | $11^{\text {th }}$ Nov 2018 | Mr. Gowtham | PO9 |
|  |  |  |  |  |  |  |

## C. COURSE ASSESSMENT

## 1. Course Coverage

Assessment of learning outcomes for Internal and end semester evaluation. Distinct assignment for each student. 1 Assignment per chapter per student. 1 seminar per test per student.

| Mod ules | Title | Teach. Hours | No. of question in Exam |  |  |  |  |  | CO | Levels |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  | CIA-1 | CIA-2 | CIA-3 | Asg | $\begin{gathered} \text { Extra } \\ \text { Asg } \\ \hline \end{gathered}$ | SEE |  |  |
| 1 | Stresses and Strain | 10 | 2 | - | - | 1 | 1 | 2 | CO1, CO2 | L2, L3 |
| 2 | Analysis of Stress and Strain and Cylinders | 10 | 2 |  | - | 1 | 1 | 2 | CO3, CO4 | L3 |
| 4 | Torsion and Theories of Failure | 10 | - | 2 | - | 1 | 1 | 2 | CO5, CO6 | L3 |
| 5 | Columns and strain Energy | 10 | - | 2 | - | 1 | 1 | 2 | C07, C08 | L3 |
| 3 | Shear Force and Bending Moment and stresses in beams | 10 | - | - | 4 | 1 | 1 | 2 | CO9, CO10 | L3 |
| - | Total | 50 | 4 | 4 | 4 | 5 | 5 | 10 | - | - |

## 2. Continuous Internal Assessment (CIA)

Assessment of learning outcomes for Internal exams. Blooms Level in last column shall match with A.2.

| Mod <br> ules | Evaluation | Weightage in <br> Marks | CO | Levels |
| :---: | :--- | :---: | :---: | :---: |
| 1,2 | CIA Exam -1 | 30 | CO1, CO2, CO3,Co4 | L2,L3,L3,L3 |
| 4,5 | CIA Exam -2 | 30 | CO5, CO6, CO7, C08 | L3,L3,L3,L3 |
| 3 | CIA Exam -3 | 30 | CO9, CO10 | L3,L3 |
|  |  |  |  |  |
| 1,2 | Assignment -1 | 10 | CO1, CO2, CO3,Co4 | L2,L3,L3,L3 |

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| 4,5 | Assignment -2 | 10 | CO5, CO6, CO7, C08 | L3,L3,L3,L3 |
| :---: | :--- | :---: | :---: | :---: |
| 3 | Assignment -3 | 10 | CO9, CO10 | L3,L3 |
|  |  |  |  |  |
| 1,2 | Seminar -1 |  | - | - |
| 4,5 | Seminar -2 |  | - | - |
| 3 | Seminar -3 |  | - | - |
|  |  |  | - | - |
| 1,2 Quiz - 1 |  | - | - |  |
| 4,5 Quiz -2 |  | - | - |  |
| 3 Quiz -3 |  | - | - |  |
| $1-5$ | Other Activities - Mini Project | - | - | - |
|  | Final CIA Marks | $\mathbf{4 0}$ | - |  |

## D1. TEACHING PLAN - 1

## Module - 1

| Title: | Stress \& Strain | Appr Time: | 10 Hrs |
| :---: | :---: | :---: | :---: |
| a | Course Outcomes | CO | Blooms |
| - | At the end of the topic the student should be able to . . . | - | Level |
| 1 | Understand simple, thermal stresses \& strains and their relations | CO1 | L2 |
| 2 | Determine the stress, strain and deformation in bars with varying cross section and temperature change | CO2 | L3 |
| b | Course Schedule | - | - |
| Class No | Portion covered per hour | - | - |
| 1 | Introduction to Stress \& Strain, Hooke's law | C01 | L2 |
| 2 | Calculation of stresses in straight bar \& stepped bar | C01 | L3 |
| 3 | Calculation of stresses in tapered sections | C01 | L3 |
| 4 | Composite sections \& problems | C01 | L3 |
| 5 | Stresses due to temperature changes | C01 | L3 |
| 6 | Shear stress \& strain, | C01 | L2 |
| 7 | Lateral strain, Poisson's ratio, Generalized Hooke's law | C02 | L2 |
| 8 | Bulk modulus, relation between elastic constants | C02 | L2 |
| 9 | Problems | C02 | L3 |
| 10 | Problems | C02 | L3 |
|  |  |  |  |
| c | Application Areas | - | - |
| - | Students should be able employ / apply the Module learning's to . . . | - | - |
| 1 | Stress and strain to find an application in analyzing as structure | CO1 | L2 |
| 2 | Structural analysis | CO2 | L3 |
|  |  |  |  |
| d | Review Questions | - | - |
| - | The attainment of the module learning assessed through following questions | - | - |
| 1 | Define I) Hooke's law (ii) Poisson's ratio (iii) Volumetric strain (iv) Modulus of rigidity v) | CO 1 | L1 |


|  | Normal stress vi) Shear stress vii) Factor of safety viii) Bulk Modulus ix) Modulus of elasticity x) Elasticity xi) plasticity xii) Resilience xiii) Toughness xvi) stiffness, |  |  |
| :---: | :---: | :---: | :---: |
| 2 | The tensile test was conducted on mild steel bar. The following data was obtained from the test, Diameter of steel bar $=16 \mathrm{~mm}$, Load at proportionality limit $=72 \mathrm{KN}$, Load at failure $=80 \mathrm{KN}$, diameter of the rod at failure $=12 \mathrm{~mm}$, gauge length of the bar $=80 \mathrm{~mm}$, extension at load of $60 \mathrm{KN}=0.115 \mathrm{~mm}$, Final gauge length of bar $=104 \mathrm{~mm}$. Determine (I) Young's modulus (ii) Proportionality limit (iii) Percentage elongation (iv) Percentage decrease in area (v) Breaking stress. | CO1 | L3 |
| 3 | Draw and explain stress-strain diagram of a mild steel specimen subjected to tension test. Mark the salient points on it. | CO1 | L2 |
| 4 | A circular rod of 100 mm diameter and 500 mm length is subjected to a tensile load of 1000 KN. Determine the I) Modulus of rigidity ii) Bulk modulus iii) Change in volume. Take Poisson's ratio $=0.30$ and $\mathrm{E}=200 \mathrm{GPa}$. | CO 2 | L3 |
| 5 | Derive a relation between modulus of elasticity and bulk modulus. | CO 2 | L2 |
| 6 | A bar of brass 25 mm diameter is enclosed in a steel tube of 50 mm external diameter and 25 mm internal diameter. The bar and the tube fastened at the ends and are 1.5 m long. Find the stresses in the two materials when the temperature raises from $30^{\circ} \mathrm{C}$ to $80^{\circ} \mathrm{C}$ | CO 2 | L3 |
| 7 | Derive an expression for total extension of tapering circular bar cross section of diameter 'D' and ' $d$ ' when subjected to an axial pull of load $P$ | CO1 | L2 |
| 8 | Derive an expression for extension of bar due to its self weight only having area A and length $L$ suspended from its top. | CO1 | L3 |
| 9 | Derive an expression for volumetric strain of rectangular bar subjected to normal stress along its axis. | CO1 | L2 |
| 10 | Derive the relationship between modulus of rigidity and Young's modulus of elasticity and define elastic constants. | CO1 | L2 |
| 11 | The modulus of rigidity for a material is 51 GPa . A 10 mm diameter rod of the material was subjected to an axial load of 10 KN and the change in diameter was observed to be 3 x $10^{-3} \mathrm{~mm}$. Calculate the Poisson's ratio and the modulus of elasticity. | CO 2 | L3 |
| 12 | A concrete column is of square section with 250 mm size and is reinforced with 08 steel bars of 16 mm diameter. The member supports an axial load of 270 KN . Evaluate the stresses in steel and concrete assuming a modular ratio as 18 | CO 2 | L3 |
| 13 | Derive an expression for the deformation of tapering circular bar subjected to axial force. | CO1 | L2 |
| 14 | When a bar of 25 mm diameter is subjected to pull of 61 KN , the extension on a 50 mm gauge length is 0.1 mm and decrease in diameter is 0.013 mm . Calculate the values of elastic constants E, G, K and $\mu$. | CO 2 | L3 |
| 15 | A compound bar is made up of a central aluminium plate 24 mm wide and 6 mm thick to which steel plates of 24 mm wide and 9 mm thick are connected rigidly on each side. The length of compound bar at temperature $20^{\circ} \mathrm{C}$ is 100 mm . If the temperature of the whole assembly is raised by $60^{\circ} \mathrm{C}$, determine the stress in each of the material. If at the new temperature a compressive load of 20 kN is applied to the composite bar. What are the final stresses in steel and aluminum? Given $\mathrm{Es}=2 \times 10^{5} \mathrm{~N} / \mathrm{mm}^{2}, \mathrm{E}_{\mathrm{A}}=-2 \mathrm{X} 10^{5}$ $\mathrm{N} / \mathrm{mm}^{2} \quad a_{\mathrm{s}}=12 \times 10^{-6}{ }^{\circ} \mathrm{C}$ and $a_{\mathrm{A}}=23 \times 10^{-6} /{ }^{\circ} \mathrm{C}$. | CO 2 | L3 |
| 16 | A composite bar is rigidly fitted at the supports A and B as shown in Fig. Determine the reactions at the supports when temperature rises by $20^{\circ}$. Take $\mathrm{EA}=70 \mathrm{GPa}$, Es $=200$ CPA, $a_{\mathrm{A}}=11 \times 10-6 /{ }^{\circ} \mathrm{C}$ and $a_{\mathrm{s}}=12 \times 10-61^{\prime \prime} \mathrm{C}$ | CO 2 | L3 |
| e | Experiences | - | - |
| 1 |  | CO1 | L2 |
| 2 |  |  |  |
| 3 |  |  |  |
| 4 |  | CO 2 | L2 |
| 5 |  |  |  |

## Module - 2

| Title: | Analysis of Stress and Strain and Cylinders | Appr Time: | 10 Hrs |
| :---: | :---: | :---: | :---: |
| a | Course Outcomes | CO | Blooms |
| - | At the end of the topic the student should be able to . . . | - | Level |
| 1 | Determine principal stresses and planes using analytical and graphical method. | CO3 | L4 |
| 2 | Determine stress distribution In thick and thin cylinder | CO4 | L3 |
| b | Course Schedule | - | - |
| Class No | Portion covered per hour | - | - |
| 11 | Plane stress, stresses on inclined planes | CO3 | L2 |
| 12 | Principal stresses and maximum shear stress principal angles | CO3 | L2 |
| 13 | Shear stresses on principal planes, Maximum shear stress | CO3 | L3 |
| 14 | Mohr's circle for plane stress conditions. | CO3 | L3 |
| 15 | Problems on Mohr's Circle | CO3 | L3 |
| 16 | Thin Cylinder Hoop's stress Maximum shear stress | CO4 | L2 |
| 17 | Circumferential and longitudinal strains | CO4 | L2 |
| 18 | Problems on thin cylinder | CO4 | L3 |
| 19 | Thick cylinders Lames equation | CO4 | L2 |
| 20 | Problems on thick cylinder | CO4 | L3 |
|  |  |  |  |
| c | Application Areas | - | - |
| - | Students should be able employ / apply the Module learnings to . . . | - | - |
| 1 | Combined loading in structural and machine parts | CO3 | L3 |
| 2 | Pressure vessels | CO4 | L3 |
|  |  |  |  |
| d | Review Questions | - | - |
| - | The attainment of the module learning assessed through following questions | - | - |
| 17 | A tension member is formed by connecting two wooden scantling each $75 \mathrm{~mm} \times 150 \mathrm{~mm}$ at their ends, which are cut at an angle of $60^{\circ}$ as shown in Fig.15. The member is subjected to a pull F. Find the safe value of F, if the permissible normal and shear stress in the glue are $1.5 \mathrm{~N} / \mathrm{mm}^{2}$ and $1 \mathrm{~N} / \mathrm{mm}^{2}$ respectively. | CO3 | L3 |
| 18 | A point in a strained material is subjected to a tensile stress of $500 \mathrm{~N} / \mathrm{mm}^{2}$ and $300 \mathrm{~N} / \mathrm{mm}^{2}$ in two mutual perpendicular planes. Calculate the normal, tangential, resultant stresses and its obliquity on a plane making an angle of $30^{\circ}$ with the axis of the second stress. Also find the maximum shear stress. | CO4 | L3 |
| 19 | A point in a plate grinder is subjected to a horizontal tensile stress of $100 \mathrm{~N} / \mathrm{mm}^{2}$ and vertical shear stress of $60 \mathrm{~N} / \mathrm{mm}^{2}$. Find the magnitude of principal stresses and its location. | CO3 | L3 |
| 20 | The state of stress in a two dimensionally stressed body is as shown in Fig. Determine (I) Normal stress on maximum shear stress on plane AC. ii) Principal stresses, principal planes and maximum shear stress and its location iii) Normal stress on maximum shear stress plane. iv) Verify answers by Mohr's circle method. | CO3 | L2 |


|  |  |  |  |
| :---: | :---: | :---: | :---: |
| 21 | An element is subjected to stresses as shown in Fig. determine I) Principal stresses and their directions.. ii) Normal and tangential stress on plane AC | CO3 | L3 |
| 22 | Direct stresses of $120 \mathrm{~N} / \mathrm{mm}^{2}$ tensile and $90 \mathrm{~N} / \mathrm{mm}^{2}$ compression exist on two perpendicular planes at a certain point in a body. They are also accompanied by shear stress on the plane. The greatest principal stress at the point due to these is $150 \mathrm{~N} / \mathrm{mm}^{2}$. Determine (I) Magnitude of shearing stress on the two planes. (ii) Maximum shearing stress at the point. | CO3 | L2 |
| 23 | The state of stress at a point in a strained material is as shown in Fig. Q3 (a). Determine : (i) The magnitude of principal stresses. (ii) The direction of principal stresses and (iii) The magnitude of the maximum shear stress and its direction. Indicate all the planes by a sketch. | CO3 | L3 |
| 24 | A point in a strained material in subjected to a tensile stress of $500 \mathrm{~N} / \mathrm{mm}$ and $300 \mathrm{~N} / \mathrm{mm}^{2}$ in a two mutually perpendicular planes and also these planes carries a shear stress of $100 \mathrm{~N} / \mathrm{mm}^{2}$. Calculate the normal, tangential, resultant stresses on a plane making an angle of $30^{\circ} \mathrm{C}$ with the vertical axis (x-plane). Also find principal stresses. | CO3 | L3 |
| 25 | An element is subjected to stresses as shown in Fig. Determine: i) Principal stresses and their directions ii) Normal and tangential stress on plane AC | CO3 | L3 |
| 26 | At a Certain point in a strained material the values of normal stresses across two planes at right angles to each other are 80 MPa , both tensile and there is a shear stress of 32 MPa cw on the plane carrying 80 MPa stresses across the planes. Determine (I) Maximum and minimum stresses and locate their planes. (ii) Maximum shear stress and specify its plane. (iii) Normal stress on maximum shear stress plane. (iv) Verify the answer by Mohr's circle method. | CO3 | L3 |
| 27 | Obtain an expression for volumetric strain in case of thin cylindrical shell of internal diameter ' $d$ ' subjected to internal pressure ' $p$ '. | CO4 | L2 |
| 28 | Derive Lame's equation for radial and hoop stress in case of thick cylinders. | CO4 | L2 |


| 29 | For a thin cylinder subjected to internal pressure ' p ' prove that the circumferential stress $=$ $\frac{p d}{2 t}$ and longitudinal stress $=\frac{p d}{4 t}$ where ' d ' is internal diameter and ' t ' is thickness of wall. | CO 4 | L2 |
| :---: | :---: | :---: | :---: |
| 30 | The maximum stress permitted in a thick cylinder radii 200 mm and 300 mm is 16 $\mathrm{N} / \mathrm{mm}^{2}$. If the internal pressure is $12 \mathrm{~N} / \mathrm{mm}^{2}$, what external pressure can be applied? Plot curves showing the variations of hoop and radial stresses through the material. | CO4 | L3 |
| 31 | A thick metallic cylindrical shell is 150 mm internal diameter is required to with stand an internal pressure of 8 MPa . Find the necessary thickness of shell if permissible tensile stress in the section is 20 MPa . | CO 4 | L3 |
| 32 | A Cylindrical pressure vessel has inner and outer diameter 340 mm and 440 mm respectively. Taking allowable stress for the material of the cylinder as $50 \mathrm{~N} / \mathrm{mm}^{2}$, determine (I) Maximum value of external pressure that can be applied (ii) Circumferential stress induced in the outer surface. | CO4 | L3 |
| 33 | A Cylindrical pressure vessel of 250 mm inner radius Is subjected to internal pressure of $10 \mathrm{~N} / \mathrm{mm}^{2}$ allowable stress for the material of the cylinder is $40 \mathrm{~N} / \mathrm{mm}^{2}$. Determine (I) Wall thickness of cylinder (ii) percentage error involved when the thickness is calculated based on thin cylinder. | CO 4 | L3 |
| 34 | A thin cylindrical shell of 1200 mm diameter, 15 mm thick and 6 m long is subjected to internal fluid pressure of $2.5 \mathrm{~N} / \mathrm{mm}^{2} \mathrm{E}=2 \times 10^{5} \mathrm{~N} / \mathrm{mm}^{2}$, $\frac{1}{m}=0.3$. Calculate (I) Change in diameter, (ii) change in length, (iii) change in volume | CO4 | L3 |
| 35 | A cylindrical pipe of diameter 2 m and wall thickness 20 mm is subjected to an internal fluid pressure of $1.5 \mathrm{~N} / \mathrm{mm}^{2}$, Determine (I) Longitudinal stress and (ii) Circumferential stress developed in the pipe material. | CO4 | L3 |
| 36 | A thin cylindrical shell 2 m long has 200 mm internal diameter and thickness of metal 10 mm . It is filled completely with a fluid at atmospheric pressure. If an additional 25000 $\mathrm{mm}^{3}$ fluid is pumped in find the pressure developed and hoop stress developed. Also find the change in diameter and length. $\mathrm{E}=2 \times 10^{5} \mathrm{~N} / \mathrm{mm}^{2}$ and $\frac{1}{m}=0.3$ | CO 4 | L3 |
|  |  |  |  |
| e | Experiences | - | - |
| 1 |  | CO3 | L2 |
| 2 |  |  |  |
| 3 |  |  |  |
| 4 |  | CO4 | L2 |
| 5 |  |  |  |

## E1. CIA EXAM - 1

a. Model Question Paper - 1

| Crs |  | 18ME32 | Sem: | 3 | Marks: | 30 | Time | 90 m | minutes |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Course: |  | Mechanics of Materials |  |  |  |  |  |  |  |  |  |
|  |  | Note: Answer all questions, each carry equal marks. Module : 1,2 |  |  |  |  |  |  | Marks | CO | Level |
| 1 | a | Derive the relationship between young's Modulus, Bulk Modulus, and Shear Modulus |  |  |  |  |  |  | 6 | CO 2 | L2 |
|  | b | The tensile test was conducted on mild steel bar. The following data was obtained from the test, Diameter of steel bar $=16 \mathrm{~mm}$, Load at proportionality limit $=72 \mathrm{KN}$ , Load at failure $=80 \mathrm{KN}$, diameter of the rod at failure $=12 \mathrm{~mm}$, gauge length of the bar $=80 \mathrm{~mm}$, extension at load of $60 \mathrm{KN}=0.115 \mathrm{~mm}$, Final gauge length of bar $=$ 104 mm. Determine (I) Young's modulus (ii) Proportionality limit (iii) Percentage elongation (iv) Percentage decrease in area (v) Breaking stress. |  |  |  |  |  |  | 9 | CO1 | L3 |
| 2 | a | Derive an expression for total extension of tapering circular bar cross section of diameter 'D' and 'd' when subjected to an axial pull of load P |  |  |  |  |  |  | 5 | CO1 | L2 |
|  | b | A concrete column is of square section with 250 mm size and is reinforced with 08 steel bars of 16 mm diameter. The member supports an axial load of 270 KN . Evaluate the stresses in steel and concrete assuming a modular ratio as 18 |  |  |  |  |  |  | 10 | CO2 | L3 |
| 3 | a | Direct stresses of $120 \mathrm{~N} / \mathrm{mm}^{2}$ tensile and $90 \mathrm{~N} / \mathrm{mm}^{2}$ compression exist on two perpendicular planes at a certain point in a body. They are also accompanied by shear stress on the plane. The greatest principal stress at the point due to these is 150 $\mathrm{N} / \mathrm{mm}^{2}$. Determine (I) Magnitude of shearing stress on the two planes. (ii) Maximum shearing stress at the point. |  |  |  |  |  |  | 7 | CO3 | L3 |
|  | b | Obtain an expression for volumetric strain in case of thin cylindrical shell of internal diameter ' $d$ ' subjected to internal pressure ' $p$ '. |  |  |  |  |  |  | 5 | CO4 | L2 |
| 4 | a | At a Certain point in a strained material the values of normal stresses across two planes at right angles to each other are 80 MPa , both tensile and there is a shear stress of 32 MPa cw on the plane carrying 80 MPa stresses across the planes. Determine (I) Maximum and minimum stresses and locate their planes. (ii) Maximum shear stress and specify its plane. (iii) Normal stress on maximum shear stress plane. (iv) Verify the answer by Mohr's circle method. |  |  |  |  |  |  | 7 | CO3 | L3 |
|  | b | A Cylindrical pressure vessel of 250 mm inner radius Is subjected to internal pressure of $10 \mathrm{~N} / \mathrm{mm}^{2}$ allowable stress for the material of the cylinder is $40 \mathrm{~N} / \mathrm{mm}^{2}$. Determine (I) Wall thickness of cylinder (ii) percentage error involved when the thickness is calculated based on thin cylinder. |  |  |  |  |  |  | 8 | CO4 | L3 |

## b. Assignment -1

Note: A distinct assignment to be assigned to each student.

| Model Assignment Questions |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Crs Code: | 18ME32 | Sem: | 3 | Marks: | 10 | Time: | $90-120$ minutes |  |  |
| Course: | Mechanics of Materials |  |  |  | Module : 1, 2 |  |  |  |  |
| Note: Each student to answer 2-3 assignments. Each assignment carries equal mark. |  |  |  |  |  |  |  |  |  |
| SNo | USN |  |  | gnment D | ripti |  | Marks | CO | Level |


| 1 | Define I) Hooke's law (ii) Poisson's ratio (iii) Volumetric strain (iv) Modulus of rigidity v) Normal stress vi) Shear stress vii) Factor of safety viii) Bulk Modulus ix) Modulus of elasticity x) Elasticity xi) plasticity xii) Resilience xiii) Toughness xvi) stiffness, | 10 | CO1 | L1 |
| :---: | :---: | :---: | :---: | :---: |
| 2 | The tensile test was conducted on mild steel bar. The following data was obtained from the test, Diameter of steel bar $=16 \mathrm{~mm}$, Load at proportionality limit $=72 \mathrm{KN}$, Load at failure $=80 \mathrm{KN}$, diameter of the rod at failure $=12 \mathrm{~mm}$, gauge length of the bar $=80 \mathrm{~mm}$, extension at load of $60 \mathrm{KN}=0.115 \mathrm{~mm}$, Final gauge length of bar $=104 \mathrm{~mm}$. Determine (I) Young's modulus (ii) Proportionality limit (iii) Percentage elongation (iv) Percentage decrease in area (v) Breaking stress. | 10 | CO1 | L3 |
| 3 | Draw and explain stress-strain diagram of a mild steel specimen subjected to tension test. Mark the salient points on it. | 10 | CO1 | L2 |
| 4 | A circular rod of 100 mm diameter and 500 mm length is subjected to a tensile load of 1000 KN . Determine the I) Modulus of rigidity ii) Bulk modulus iii) Change in volume. Take Poisson's ratio $=0.30$ and $\mathrm{E}=$ 200 GPa . | 10 | CO2 | L3 |
| 5 | Derive a relation between modulus of elasticity and bulk modulus. | 10 | CO2 | L2 |
| 6 | A bar of brass 25 mm diameter is enclosed in a steel tube of 50 mm external diameter and 25 mm internal diameter. The bar and the tube fastened at the ends and are 1.5 m long. Find the stresses in the two materials when the temperature raises from $30^{\circ} \mathrm{C}$ to $80^{\circ} \mathrm{C}$ | 10 | CO2 | L3 |
| 7 | Derive an expression for total extension of tapering circular bar cross section of diameter 'D' and 'd' when subjected to an axial pull of load P | 10 | CO1 | L2 |
| 8 | Derive an expression for extension of bar due to its self weight only having area A and length L suspended from its top. | 10 | CO1 | L3 |
| 9 | Derive an expression for volumetric strain of rectangular bar subjected to normal stress along its axis. | 10 | CO1 | L2 |
| 10 | Derive the relationship between modulus of rigidity and Young's modulus of elasticity and define elastic constants. | 10 | CO1 | L2 |
| 11 | The modulus of rigidity for a material is 51 GPa . A 10 mm diameter rod of the material was subjected to an axial load of 10 KN and the change in diameter was observed to be $3 \times 10^{-3} \mathrm{~mm}$. Calculate the Poisson's ratio and the modulus of elasticity. | 10 | CO2 | L3 |
| 12 | A concrete column is of square section with 250 mm size and is reinforced with 08 steel bars of 16 mm diameter. The member supports an axial load of 270 KN . Evaluate the stresses in steel and concrete assuming a modular ratio as 18 | 10 | CO2 | L3 |
| 13 | Derive an expression for the deformation of tapering circular bar subjected to axial force. | 10 | CO1 | L2 |
| 14 | When a bar of 25 mm diameter is subjected to pull of 61 KN , the extension on a 50 mm gauge length is 0.1 mm and decrease in diameter is 0.013 mm . Calculate the values of elastic constants $\mathrm{E}, \mathrm{G}, \mathrm{K}$ and $\mu$. | 10 | CO2 | L3 |
| 15 | Derive the relationship between Young's Modulus and Shear Modulus | 10 | CO2 | L2 |
| 16 | A tension member is formed by connecting two wooden scantling each $75 \mathrm{~mm} \times 150 \mathrm{~mm}$ at their ends, which are cut at an angle of $60^{\circ}$ as shown in Fig.15. The member is subjected to a pull F. Find the safe value of F , if the permissible normal and shear stress in the glue are $1.5 \mathrm{~N} / \mathrm{mm}^{2}$ and $1 \mathrm{~N} / \mathrm{mm}^{2}$ respectively. | 10 | CO3 | L3 |
| 17 | A point in a strained material is subjected to a tensile stress of 500 $\mathrm{N} / \mathrm{mm}^{2}$ and $300 \mathrm{~N} / \mathrm{mm}^{2}$ in two mutual perpendicular planes. Calculate the normal, tangential, resultant stresses and its obliquity on a plane making an angle of $30^{\circ}$ with the axis of the second stress. Also find the maximum shear stress. | 10 | CO4 | L3 |
| 18 | A point in a plate grinder is subjected to a horizontal tensile stress of $100 \mathrm{~N} / \mathrm{mm}^{2}$ and vertical shear stress of $60 \mathrm{~N} / \mathrm{mm}^{2}$. Find the magnitude of principal stresses and its location. | 10 | CO3 | L3 |
| 19 | The state of stress in a two dimensionally stressed body is as shown in Fig. Determine (I) Normal stress on maximum shear stress on plane AC. ii) Principal stresses, principal planes and maximum shear stress | 10 | CO3 | L2 |

|  | and its location iii) Normal stress on maximum shear stress plane. iv) Verify answers by Mohr's circle method. |  |  |  |
| :---: | :---: | :---: | :---: | :---: |
| 20 | An element is subjected to stresses as shown in Fig. determine I) Principal stresses and their directions.. ii) Normal and tangential stress on plane AC | 10 | CO3 | L3 |
| 21 | Direct stresses of $120 \mathrm{~N} / \mathrm{mm}^{2}$ tensile and $90 \mathrm{~N} / \mathrm{mm}^{2}$ compression exist on two perpendicular planes at a certain point in a body. They are also accompanied by shear stress on the plane. The greatest principal stress at the point due to these is $150 \mathrm{~N} / \mathrm{mm}^{2}$. Determine (I) Magnitude of shearing stress on the two planes. (ii) Maximum shearing stress at the point. | 10 | CO3 | L2 |
| 22 | The state of stress at a point in a strained material is as shown in Fig. Q3 (a). Determine : (i) The magnitude of principal stresses. (ii) The direction of principal stresses and (iii) The magnitude of the maximum shear stress and its direction. Indicate all the planes by a sketch. | 10 | CO3 | L3 |
| 23 | A point in a strained material in subjected to a tensile stress of $500 \mathrm{~N} / \mathrm{mm}^{`}$ and $300 \mathrm{~N} / \mathrm{mm}^{2}$ in a two mutually perpendicular planes and also these planes carries a shear stress of $100 \mathrm{~N} / \mathrm{mm}^{2}$. Calculate the normal, tangential, resultant stresses on a plane making an angle of $30^{\circ} \mathrm{C}$ with the vertical axis (x-plane). Also find principal stresses. | 10 | CO3 | L3 |
| 24 | An element is subjected to stresses as shown in Fig. Determine: i) Principal stresses and their directions ii) Normal and tangential stress on plane AC | 10 | CO3 | L3 |
| 25 | At a Certain point in a strained material the values of normal stresses across two planes at right angles to each other are 80 MPa , both tensile and there is a shear stress of 32 MPa cw on the plane carrying 80 MPa stresses across the planes. Determine (I) Maximum and minimum stresses and locate their planes. (ii) Maximum shear stress and specify its plane. (iii) Normal stress on maximum shear stress plane. (iv) Verify the answer by Mohr's circle method. | 10 | CO3 | L3 |
| 26 | Obtain an expression for volumetric strain in case of thin cylindrical shell of internal diameter ' $d$ ' subjected to internal pressure ' $p$ '. | 10 | CO4 | L2 |

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| 27 | Derive Lame's equation for radial and hoop stress in case of thick cylinders. | 10 | CO 4 | L2 |
| :---: | :---: | :---: | :---: | :---: |
| 28 | For a thin cylinder subjected to internal pressure ' p ' prove that the circumferential stress $=\frac{p d}{2 t}$ and longitudinal stress $=\frac{p d}{4 t}$ where ' d ' is internal diameter and ' $t$ ' is thickness of wall. | 10 | CO4 | L2 |
| 29 | The maximum stress permitted in a thick cylinder radii 200 mm and 300 mm is $16 \mathrm{~N} / \mathrm{mm}^{2}$. If the internal pressure is $12 \mathrm{~N} / \mathrm{mm}^{2}$, what external pressure can be applied? Plot curves showing the variations of hoop and radial stresses through the material. | 10 | CO4 | L3 |
| 30 | A thick metallic cylindrical shell is 150 mm internal diameter is required to with stand an internal pressure of 8 MPa . Find the necessary thickness of shell if permissible tensile stress in the section is 20 MPa . | 10 | CO4 | L3 |
| 31 | A Cylindrical pressure vessel has inner and outer diameter 340 mm and 440 mm respectively. Taking allowable stress for the material of the cylinder as $50 \mathrm{~N} / \mathrm{mm}^{2}$, determine (I) Maximum value of external pressure that can be applied (ii) Circumferential stress induced in the outer surface. | 10 | CO4 | L3 |
| 32 | A Cylindrical pressure vessel of 250 mm inner radius Is subjected to internal pressure of $10 \mathrm{~N} / \mathrm{mm}^{2}$ allowable stress for the material of the cylinder is $40 \mathrm{~N} / \mathrm{mm}^{2}$. Determine (I) Wall thickness of cylinder (ii) percentage error involved when the thickness is calculated based on thin cylinder. | 10 | CO4 | L3 |
| 33 | A thin cylindrical shell of 1200 mm diameter, 15 mm thick and 6 m long is subjected to internal fluid pressure of $2.5 \mathrm{~N} / \mathrm{mm}^{2} \mathrm{E}=2 \times 10^{5}$ $\mathrm{N} / \mathrm{mm}^{2}$, <br> $\frac{1}{m}=0.3$. Calculate (I) Change in diameter, (ii) change in length, (iii) change in volume | 10 | CO4 | L3 |
| 34 | A cylindrical pipe of diameter 2 m and wall thickness 20 mm is subjected to an internal fluid pressure of $1.5 \mathrm{~N} / \mathrm{mm}^{2}$, Determine (I) Longitudinal stress and (ii) Circumferential stress developed in the pipe material. | 10 | CO4 | L3 |
| 35 | A thin cylindrical shell 2 m long has 200 mm internal diameter and thickness of metal 10 mm . It is filled completely with a fluid at atmospheric pressure. If an additional $25000 \mathrm{~mm}^{3}$ fluid is pumped in find the pressure developed and hoop stress developed. Also find the change in diameter and length. $\mathrm{E}=2 \times 10^{5} \mathrm{~N} / \mathrm{mm}^{2}$ and $\frac{1}{m}=0.3$ | 10 | CO4 | L3 |
| 36 | A room temperature the gap between bar A and bar B shown in Fig is 0.25 mm . What are the stresses induced in the bars, if temperature rise is $35^{\circ} \mathrm{C}$. Given, $\mathrm{A}_{\mathrm{A}}=1000 \mathrm{~mm}^{2} ; \mathrm{A}_{\mathrm{B}}=800 \mathrm{~mm}^{2} ; \mathrm{E}_{\mathrm{A}}=200 \mathrm{GPa} ; \mathrm{E}_{\mathrm{B}}=100$ GPa; $\alpha_{a}=12 \times 10^{-6}$ per ${ }^{\circ} \mathrm{C} ; \alpha_{b}=23 \times 10^{-6}$ per ${ }^{\circ} \mathrm{C} ; \mathrm{L}_{\mathrm{A}}=400 \mathrm{~mm} ; \mathrm{L}_{\mathrm{B}}=$ 300 mm ; | 10 | CO2 | L3 |
| 37 | A composite bar shown in Fig. Is 0.25 mm short of distance between the rigid supports at room temperature. What is the maximum temperature rise which will not produce stresses in the bar? Find stresses induced when temperature rise is $400^{\circ} \mathrm{C}$. Given $\alpha_{s}=12 \times 10^{-6}$ per ${ }^{\circ} \mathrm{C} ; \alpha_{c}=17.5 \times 10^{-6}$ per ${ }^{\circ} \mathrm{C} ; \mathrm{E}_{\mathrm{s}}=2 \times 10^{5} \mathrm{~N} / \mathrm{mm}^{2} ; \mathrm{E}_{\mathrm{c}}=1.2 \times 10^{5}$ $\mathrm{N} / \mathrm{mm}^{2} ; \mathrm{A}_{\mathrm{s}}: \mathrm{A}_{\mathrm{c}}=4: 3$ | 10 | CO2 | L3 |



|  | find the alteration or change in length. $\mathrm{E}=2 \times 10^{5} \mathrm{~N} / \mathrm{mm}^{2}$ and poisson's ratio $=0.25$ |  |  |  |
| :---: | :---: | :---: | :---: | :---: |
| 48 | A bar of rectangular cross section $20 \mathrm{~mm} \times 50 \mathrm{~mm}$ is 400 mm long and subjected to an axial tensile load of 80 KN . If the modulus of elasticity and modulus of rigidity of the material of bar are $1 \times 10^{5} \mathrm{~N} / \mathrm{mm}^{2}$ and $0.4 \times 10^{5} \mathrm{~N} / \mathrm{mm}^{2}$, determine Bulk modulus, change in dimensions and volume. | 10 | CO 2 | L3 |
| 49 | Calculate the modulus of rigidity and bulk modulus of a cylindrical bar of diameter 25 mm and length 1.6 m , if the longitudinal strain in the bar during tensile test is four times the lateral strain. Also find the change in volume, when the bar is subjected to hydrostatic pressure of 100 $\mathrm{N} / \mathrm{mm}^{2}$. $\mathrm{E}=1 \times 10^{5} \mathrm{~N} / \mathrm{mm}^{2}$. | 10 | CO 2 | L3 |
| 50 | A bar of 20 mm diameter is subjected to a pull of 50 KN . The measured extension on gauge length of 250 mm is 0.12 mm and change in diameter is 0.00375 mm . Determine (I) Young's modulus (ii) Poisson's ratio (iii) Bulk modulus (iv) Modulus of rigidity. | 10 | CO 2 | L3 |

## D2. TEACHING PLAN - 2

## Module - 4

| Title: | Torsion and Theories of failures | Appr Time: | 10 Hrs |
| :---: | :---: | :---: | :---: |
| a | Course Outcomes | CO | Blooms |
| - | At the end of the topic the student should be able to . . . | - | Level |
| 1 | Determine, torsional strength, rigidity, flexibility \& dimension of shaft | CO5 | L3 |
| 2 | Determine dimension of bars using theories of failure | CO6 | L3 |
|  |  |  |  |
| b | Course Schedule |  |  |
| Class No | Portion covered per hour | - | - |
| 21 | Circular solid and hallow shafts | CO5 | L2 |
| 22 | Torsional moment of resistance | CO5 | L2 |
| 23 | Power transmission of straight and stepped shafts | CO5 | L2 |
| 24 | Twist in shaft sections, thin tubular section, thin walled sections | CO5 | L3 |
| 25 | Numerical | CO5 | L3 |
| 26 | Maximum Principal stress theory | CO6 | L2 |
| 27 | Numerical | CO6 | L3 |
| 28 | Numerical | C06 | L3 |
| 29 | Maximum Shear stress theory | CO6 | L2 |
| 30 | Numerical | CO6 | L3 |
|  |  |  |  |
| c | Application Areas | - | - |
| - | Students should be able employ / apply the Module learnings to . . . | - | - |
| 1 | Design of shaft | CO5 | L3 |
| 2 | Failure theories analysis | CO6 | L3 |
|  |  |  |  |
| d | Review Questions | - | - |
| - | The attainment of the module learning assessed through following questions | - | - |
| 37 | A solid shaft of 250 mm diameter has the same cross sectional area as the hollow shaft of the same material with inside diameter of 200 mm . Determine (I) ratio of power transmitted by the hollow shaft and the solid shaft for the same speed. (ii) Ratio of angle of twist by the hollow shaft and the solid shaft for the same maximum stress. | CO7 | L3 |
| 38 | Show that a hollow circular shaft whose inner diameter is half the outer diameter has a torsional strength equal to $15 / 16$ of that of a solid shaft of same outside diameter. | CO7 | L3 |
| 39 | A solid shaft is required to transmit 300 KW power at 100 rpm . The shear stress of the materiel must not exceed 80 MPa . (I) Determine the diameter of solid shaft (ii) If the shaft is replaced by a hollow one with diameter ratio $3: 5$ and maximum shear stress remaining unchanged, calculate the percentage saving in weight of the material. | CO5 | L3 |
| 40 | A hollow shaft has to transmit 600 KW power at 80 rpm . The maximum twisting moment | CO5 | L3 |


|  | may exceed the mean by $40 \%$.Design a suitable section, if the permissible stress is 90 $\mathrm{N} / \mathrm{mm}^{2}$ and the diameter ratio is to be 0.8 . What will be the angular twist measured over a length of 2 m , if the modulus of rigidity is 84 GPa . |  |  |
| :---: | :---: | :---: | :---: |
| 41 | Two shafts of the same material and of same lengths are subjected to the same torque, if the first shaft if of solid circular section and second shaft is hollow circular section, whose internal diameter is 0.7 times the outside diameter and the maximum stress developed in each shaft is the same, compare the weight of the shaft | CO5 | L3 |
| 42 | Define polar modulus. Find the expression for polar modulus of a solid shaft | CO5 | L2 |
| 43 | What are the assumptions made in the pure torsion. | CO5 | L2 |
| 44 | Derive the relation for a circular shaft when subjected to torsion as given below $\frac{T}{J p}=\frac{\tau}{R}=$ $\frac{G \theta}{l}$ | CO5 | L2 |
| 45 | Define the term torsional rigidity and torsional strength. | CO5 | L1 |
| 46 | What do you mean by theories of failure? What is their importance? | CO6 | L2 |
| 47 | Explain the following theories of failure.(i) Maximum normal stress theory (ii) Maximum shear stress theory. | CO6 | L2 |
| 48 | The stress induced at a critical point in a machine component made of steel are as follows : ax $=100 \mathrm{~N} / \mathrm{mm} 2$, ay $=40 \mathrm{~N} / \mathrm{mm} 2, \mathrm{tXy}=80 \mathrm{~N} / \mathrm{mm} 2$. Calculate the factor of safety by i) Maximum shear stress theory ii) Maximum normal stress theory | CO6 | L3 |
| 49 | Find the diameter of round rod subjected to a bending moment of $1.8 \mathrm{kN}-\mathrm{m}$ and a torque of $1.2 \mathrm{kN}-\mathrm{m}$, according to i) Maximum normal stress theory ii) Maximum shear stress theory. Take allowable normal stress as 120 MPa and allowable shear stress as 72 MPa . | CO6 | L3 |
| 50 | A bolt is acted upon by an axial pull of 16 kN along with a transverse shear force of 10 kN . Determine the diameter of the bolt required, using i) Max. principal stress theory ii) Max. shear stress theory iii) Max. strain theory Elastic limit in tension $=250 \mathrm{MPa}$ Factor of safety $=2.5$ Poisson's ratio $=0.3$ | CO6 | L3 |
| 51 | A plate of 45 C 8 steel ( $\sigma y t=353 \mathrm{MPa}$ ) is subjected to the following stresses $\sigma x=150 \mathrm{~N} / \mathrm{mm}^{2}, \sigma y=100 \mathrm{~N} / \mathrm{mm}^{2}$ and $\tau x y=50 \mathrm{~N} / \mathrm{mm} 2$. Find the factor of safety by i) Rankine's theory ii) Guest's theory. | CO6 | L3 |
|  |  |  |  |
| e | Experiences | - | - |
| 1 |  | CO6 | L2 |
| 2 |  |  |  |
| 3 |  |  |  |
| 4 |  | CO6 | L2 |
| 5 |  |  |  |

Module - 5

| Title: | Columns and Strain Energy | Appr Time: | 10 Hrs |
| :---: | :---: | :---: | :---: |
| a | Course Outcomes | CO | Blooms |
| - | At the end of the topic the student should be able to . . . | - | Level |
| 1 | A column or pillar in architecture and structural engineering is a structural element that transmits, through compression, | CO7 | L3 |
| 2 | The strain energy in the form of elastic deformation is mostly recoverable in the form of mechanical work. | CO8 | L3 |
| b | Course Schedule |  |  |
| Class No | Portion covered per hour | - | - |
| 31 | Buckling and stability | CO7 | L2 |
| 32 | Critical load Columns with pinned ends | CO7 | L2 |
| 33 | Columns with other support conditions | CO7 | L2 |
| 34 | Effective length of columns, secant formula for columns | C07 | L2 |
| 35 | Numerical | C07 | L3 |
| 36 | Castigliano's Theorem I and II | CO8 | L2 |
| 37 | Load deformation diagram | CO8 | L2 |
| 38 | Strain energy due to normal stresses, shear stresses modulus of resilience | CO8 | L3 |
| 39 | Strain energy due to bending and torsion. | CO8 | L3 |
| 40 | Numerical | CO8 | L3 |
|  |  |  |  |
|  |  |  |  |
| c | Application Areas | - | - |
| - | Students should be able employ / apply the Module learnings to . . . | - | - |
| 1 | Design of columns | C07 | L3 |
| 2 | Strain energy stored in the materials | CO8 | L3 |
|  |  |  |  |
| d | Review Questions | - | - |
| - | The attainment of the module learning assessed through following questions | - | - |
| 52 | A 2 m long column with hollow circular section of outer diameter 180 mm is subjected to a load of 212.5 KN . Yield stress for the column material is $340 \mathrm{~N} / \mathrm{mm}^{2}$. One end of the column is fixed and the other end is free. Determine the inner diameter, taking $\mathrm{E}=210$ GPa and FOS = 4 | CO7 | L3 |
| 53 | Determine the buckling load for a strut of ' T ' section, the flange width being 100 mm , overall depth 80 mm and both flange and web 10 mm thick. The strut is 3 m long and is hinged at both ends $\mathrm{E}=200 \mathrm{GPa}$ | CO7 | L3 |
| 54 | A column of square section is 2 m long and pinned at its ends. Young's modulus for the column material is 12 GPa and the allowable stress being 12 MPa . Determine the size of column to support a load of 200 KN safely, Using a factor of safety of 3 | CO7 | L3 |
| 55 | Find Euler's crippling load for a hollow cylindrical steel column of 38 mm external diameter and 2.5 mm wall thickness. Length of column is 2.3 m and is hinged at both ends. Also estimate Rankine's load for this column, Rankine's parameter are $335 \mathrm{~N} / \mathrm{mm}^{2}$ and 1/7500 | CO7 | L3 |
| 56 | Find the length of a mild steel rod $25 \mathrm{~mm} \times 25 \mathrm{~mm}$ which can be used as a compression member with one end fixed and the other end free to carry a working load of 40 Kn . Given FOS $=4, \alpha=\frac{1}{7500} \sigma_{c}=320 \mathrm{~N} / \mathrm{mm}^{2}$ | CO7 | L3 |
| 57 | A cast iron column 140 mm external diameter and 70 mm internal diameter is 3.5 m long. Calculate the safe axial load the column can carry if both ends are fixed. Take $\sigma_{c}=$ $600 \mathrm{MPa} ; \alpha=\frac{1}{1600}$ and factor of safety $=3$ use Rankine's formula | CO7 | L3 |
| 58 | What are the assumption and limitation of Euler's formula | CO7 | L2 |

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| 59 | Define slenderness ration and discuss the importance of slenderness ration on the compressive strength of columns. | $\mathrm{CO7}$ | L2 |
| :---: | :---: | :---: | :---: |
| 60 | What are columns, post, strut and a boom | CO7 | L2 |
| 61 | Derive the expression for Euler's critical load for a long column with both ends fixed. | CO7 | L2 |
| 62 | What is the difference between short and long columns | CO7 | L2 |
| 63 | What is meant by the effective length of a column? State the values of effective length for various column end conditions. | CO7 | L2 |
| 64 | Define the terms, strain energy, resilience and modulus of resilience. | CO8 | L2 |
| 65 | State and prove Castigliano's first theorem. | CO8 | L2 |
| 66 | State and prove Castigliano's Second theorem | CO8 | L2 |
| 67 | A 1.6 m long bar is applied an axial pull such that the maximum stress induced is 140 MPa. The larger and smaller area of cross section are $240 \mathrm{~mm}^{2}$ and $120 \mathrm{~mm}^{2}$. Determine the strain energy stored in the bar as shown in Fig 83. | CO8 | L3 |
| 68 | A bar of length 400 mm and diameter 50 mm is subjected to a tensile load of 20 KN . Find the stress, elongation and strain energy produced if the load is applied gradually. What would be the instantaneous stress and elongation if the same load is applied suddenly. Take E $=200 \mathrm{GPa}$. | CO8 | L3 |
| 69 | A weight of 2 KN falls 24 mm on to a collar fixed to a steel bar 14 mm in diameter and 5.5 m long. Determine he maximum sress induced in the bar due to impact. $\mathrm{E}=205 \mathrm{GPa}$ | CO8 | L3 |
| 70 | Calculate the strain energy stored in a bar 2 m long, 50 mm wide and 40 mm thick, when it is subjected to tensile load of 50 KN . Take E = 200 GPa . | CO8 | L3 |
| 71 | A Rectangular body 500 mm long, 100 mm wide and 50 mm thick is subjected to a shear stress of 80 MPa . Determine the strain energy stored in the body, $\mathrm{G}=85 \mathrm{GPa}$. | CO8 | L3 |
| 72 | A stepped steel rod is in two parts, one has diameter 25 mm . The box is lowered at the rate of $1 \mathrm{~m} / \mathrm{sec}$. When the freely hanging rope is 12 m , the machine gets load is brought to a stop instantaneously. Find the maximum stress in the rope and the instantaneous deformation due to jamming. Take $\mathrm{E}=200 \mathrm{GPa}$. | CO8 | L3 |
| 73 | A cantilever beam is loaded as shown in Fig. Determine the deflection and slope at the free end by using Castigliano's theorem. | CO8 | L3 |
| 74 | A shaft is in two parts as shown in Fig. Find the total strain energy stored in it Take $G=$ 85 GPa . | CO8 | L3 |
| 75 | A simple beam ACB supporting an uniform load w over the first half of the beam and a couple of moment $M_{o}$ at end B is shown in Fig. 91. Determine the strain energy $U$ stored in the beam due to the load w and the couple $\mathrm{M}_{\mathrm{o}}$ acting simultaneously. | CO8 | L3 |
| 76 | A cantilever beam of uniform cross section carries a point load at the free end. Determine strain energy and deflection at the free end. If $\mathrm{F}=200 \mathrm{kN}, \mathrm{E}=200 \mathrm{GPa}, \mathrm{L}=3 \mathrm{mt}$ and $\mathrm{I}=$ $10^{-4} \mathrm{~m}^{4}$. | CO8 | L3 |
| 77 | Two bars, each of length and of different materials are each subjected to the same tensile force $P$. The first bar has a uniform diameter ' D ' and the second bar has a diameter of $\mathrm{D} / 2$ for a length $L / 4$ and a diameter $D$ for the remaining length. Compare the strain energies of the two bars if, (i) $\mathrm{E} 1 / \mathrm{E} 2=4 / 7$ and (ii) $\mathrm{E} 1=\mathrm{E} 2$ | CO8 | L3 |
| 78 | Define strain energy and Resilience | CO8 | L1 |
| e | Experiences | - | - |
| 1 |  | CO7 | L2 |
| 2 |  |  |  |
| 3 |  |  |  |
| 4 |  | CO8 | L2 |
| 5 |  |  |  |

## E2. CIA EXAM - 2

## a. Model Question Paper - 2

| Crs Code: | 18ME32 Sem: | III | Marks: | 30 | Time: | 90 minutes |
| :---: | :--- | :--- | :--- | :--- | :--- | :---: | :---: |
| Course: | Mechanics of Materials |  |  |  |  |  |
| - | - | Note: Answer all questions, each carry equal marks. Module :4, $\mathbf{5}$ | Marks | CO | Level |  |
| 1 | a | Derive the relation for a circular shaft when subjected to torsion as given below <br> $\frac{T}{J p}=\frac{\tau}{R}=\frac{G \theta}{l}$ | 08 | CO5 | L2 |  |


|  | b | According to the theory of maximum shear stress, determine the diameter of a bolt which is subjected to an axial pull of 9 KN together with a transverse shear force of 4.5 KN. Elastic limit in tension is $225 \mathrm{~N} / \mathrm{mm} 2$, factor of safety $=3$ \& Poisson's ratio=0.3 | 07 | CO6 | L3 |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | OR |  |  |  |
| 2 | a | A solid shaft is required to transmit 300 KW power at 100 rpm . The shear stress of the materiel must not exceed 80 MPa . (I) Determine the diameter of solid shaft (ii) If the shaft is replaced by a hollow one with diameter ratio $3: 5$ and maximum shear stress remaining unchanged, calculate the percentage saving in weight of the material. | 8 | CO5 | L3 |
|  | b | Explain the following theories of failure.(i) Maximum normal stress theory (ii) Maximum shear stress theory. | 7 | CO6 | L2 |
| 3 | a | A cantilever beam of uniform cross section carries a point load at the free end. Determine strain energy and deflection at the free end. If $\mathrm{F}=200 \mathrm{kN}, \mathrm{E}=200 \mathrm{GPa}, \mathrm{L}$ $=3 \mathrm{mt}$ and $\mathrm{I}=10^{-4} \mathrm{~m}^{4}$. | 7 | CO8 | L3 |
|  | b | A cast iron column 140 mm external diameter and 70 mm internal diameter is 3.5 m long. Calculate the safe axial load the column can carry if both ends are fixed. Take $\sigma_{c}=600 \mathrm{MPa} ; \alpha=\frac{1}{1600}$ and factor of safety $=3$ use Rankine's formula | 8 | CO7 | L3 |
|  |  | OR |  |  |  |
| 4 | a | Calculate the strain energy stored in a bar 2 m long, 50 mm wide and 40 mm thick, when it is subjected to tensile load of 50 KN . Take $\mathrm{E}=200 \mathrm{GPa}$. | 8 | CO8 | L3 |
|  | b | Derive the expression for the crippling load for $\mathrm{tn}=$ he column for the both ends pinned condition | 7 | CO7 | L2 |

## b. Assignment - 2

Note: A distinct assignment to be assigned to each student.

| Model Assignment Questions |  |  |  |  |  |  |  |
| :--- | :--- | :--- | :--- | :--- | :--- | :--- | :---: |
| Crs Code: | 18ME32 | Sem: | 3 | Marks: | 10 | Time: |  |
| Course: | Mechanics of Materials |  | Module $: 4,5$ | $90-120$ minutes |  |  |  |

Note: Each student to answer 2-3 assignments. Each assignment carries equal mark.

| SNo | USN | Assignment Description | Marks | CO | Level |
| :---: | :---: | :--- | :---: | :---: | :---: |
| 1 |  | A solid shaft of 250 mm diameter has the same cross sectional area as <br> the hollow shaft of the same material with inside diameter of 200 mm. <br> Determine (I) ratio of power transmitted by the hollow shaft and the <br> solid shaft for the same speed. (ii) Ratio of angle of twist by the hollow <br> shaft and the solid shaft for the same maximum stress. | CO5 | L3 |  |
| 2 |  | Show that a hollow circular shaft whose inner diameter is half the outer | 10 | CO5 | L3 |


|  | diameter has a torsional strength equal to $15 / 16$ of that of a solid shaft of same outside diameter. |  |  |  |
| :---: | :---: | :---: | :---: | :---: |
| 3 | A solid shaft is required to transmit 300 KW power at 100 rpm . The shear stress of the materiel must not exceed 80 MPa . (I) Determine the diameter of solid shaft (ii) If the shaft is replaced by a hollow one with diameter ratio $3: 5$ and maximum shear stress remaining unchanged, calculate the percentage saving in weight of the material. | 10 | CO5 | L3 |
| 4 | A hollow shaft has to transmit 600 KW power at 80 rpm . The maximum twisting moment may exceed the mean by $40 \%$.Design a suitable section, if the permissible stress is $90 \mathrm{~N} / \mathrm{mm}^{2}$ and the diameter ratio is to be 0.8 . What will be the angular twist measured over a length of 2 m , if the modulus of rigidity is 84 GPa . | 10 | CO7 | L3 |
| 5 | Two shafts of the same material and of same lengths are subjected to the same torque, if the first shaft if of solid circular section and second shaft is hollow circular section, whose internal diameter is 0.7 times the outside diameter and the maximum stress developed in each shaft is the same, compare the weight of the shaft | 10 | CO5 | L3 |
| 6 | Define polar modulus. Find the expression for polar modulus of a solid shaft | 10 | CO5 | L2 |
| 7 | What are the assumptions made in the pure torsion. | 10 | CO5 | L2 |
| 8 | Derive the relation for a circular shaft when subjected to torsion as given below $\frac{T}{J p}=\frac{\tau}{R}=\frac{G \theta}{l}$ | 10 | CO5 | L2 |
| 9 | Define the term torsional rigidity and torsional strength. | 10 | CO5 | L2 |
| 10 | What do you mean by theories of failure? What is their importance? | 10 | CO6 | L2 |
| 11 | Explain the following theories of failure.(i) Maximum normal stress theory (ii) Maximum shear stress theory. | 10 | CO6 | L2 |
| 12 | The stress induced at a critical point in a machine component made of steel are as follows: ax $=100 \mathrm{~N} / \mathrm{mm} 2$, ay $=40 \mathrm{~N} / \mathrm{mm} 2$, $\mathrm{tXy}=$ $80 \mathrm{~N} / \mathrm{mm} 2$. Calculate the factor of safety by i) Maximum shear stress theory ii) Maximum normal stress theory | 10 | CO6 | L3 |
| 13 | Find the diameter of round rod subjected to a bending moment of 1.8 $\mathrm{kN}-\mathrm{m}$ and a torque of $1.2 \mathrm{kN}-\mathrm{m}$, according to i) Maximum normal stress theory ii) Maximum shear stress theory. Take allowable normal stress as 120 MPa and allowable shear stress as 72 MPa . | 10 | CO6 | L3 |
| 14 | A bolt is acted upon by an axial pull of 16 kN along with a transverse shear force of 10 kN . Determine the diameter of the bolt required, using i) Max. principal stress theory ii) Max. shear stress theory iii) Max. strain theory Elastic limit in tension $=250 \mathrm{MPa}$ Factor of safety $=2.5$ Poisson's ratio $=0.3$ | 10 | CO6 | L3 |
| 15 | A plate of 45 C 8 steel $(\sigma y t=353 \mathrm{MPa})$ is subjected to the following stresses $\sigma x=150 \mathrm{~N} / \mathrm{mm}^{2}, \sigma y=100 \mathrm{~N} / \mathrm{mm}^{2}$ and $\tau x y=50 \mathrm{~N} / \mathrm{mm} 2$. Find the factor of safety by i) Rankine's theory ii) Guest's theory. | 10 | CO6 | L3 |
| 16 | A 2 m long column with hollow circular section of outer diameter 180 mm is subjected to a load of 212.5 KN . Yield stress for the column material is $340 \mathrm{~N} / \mathrm{mm}^{2}$. One end of the column is fixed and the other end is free. Determine the inner diameter, taking $\mathrm{E}=210 \mathrm{GPa}$ and FOS $=4$ | 10 | CO7 | L3 |
| 17 | Determine the buckling load for a strut of ' T ' section, the flange width being 100 mm , overall depth 80 mm and both flange and web 10 mm thick. The strut is 3 m long and is hinged at both ends $\mathrm{E}=200 \mathrm{GPa}$ | 10 | CO7 | L3 |
| 18 | A column of square section is 2 m long and pinned at its ends. Young's modulus for the column material is 12 GPa and the allowable stress being 12 MPa . Determine the size of column to support a load of 200 KN safely, Using a factor of safety of 3 | 10 | CO7 | L3 |
| 19 | Find Euler's crippling load for a hollow cylindrical steel column of 38 mm external diameter and 2.5 mm wall thickness. Length of column is 2.3 m and is hinged at both ends. Also estimate Rankine's load for this column, Rankine's parameter are $335 \mathrm{~N} / \mathrm{mm}^{2}$ and $1 / 7500$ | 10 | CO7 | L3 |
| 20 | Find the length of a mild steel rod $25 \mathrm{~mm} \times 25 \mathrm{~mm}$ which can be used as a compression member with one end fixed and the other end free to | 10 | CO7 | L3 |


|  | carry a working load of 40 Kn . Given FOS $=4, \alpha=\frac{1}{7500} \sigma_{c}=$ $320 \mathrm{~N} / \mathrm{mm}^{2}$ |  |  |  |
| :---: | :---: | :---: | :---: | :---: |
| 21 | A cast iron column 140 mm external diameter and 70 mm internal diameter is 3.5 m long. Calculate the safe axial load the column can carry if both ends are fixed. Take $\sigma_{c}=600 \mathrm{MPa} ; \alpha=\frac{1}{1600}$ and factor of safety $=3$ use Rankine's formula | 10 | CO7 | L3 |
| 22 | What are the assumption and limitation of Euler's formula | 10 | CO7 | L2 |
| 23 | Define slenderness ration and discuss the importance of slenderness ration on the compressive strength of columns. | 10 | CO7 | L2 |
| 24 | What are columns, post, strut and a boom | 10 | CO7 | L2 |
| 25 | Derive the expression for Euler's critical load for a long column with both ends fixed. | 10 | CO7 | L2 |
| 26 | What is the difference between short and long columns | 10 | CO7 | L2 |
| 27 | What is meant by the effective length of a column? State the values of effective length for various column end conditions. | 10 | CO7 | L2 |
| 28 | Define the terms, strain energy, resilience and modulus of resilience. | 10 | CO8 | L2 |
| 29 | State and prove Castigliano's first theorem. | 10 | CO8 | L2 |
| 30 | State and prove Castigliano's Second theorem | 10 | CO8 | L2 |
| 31 | A 1.6 m long bar is applied an axial pull such that the maximum stress induced is 140 MPa . The larger and smaller area of cross section are $240 \mathrm{~mm}^{2}$ and $120 \mathrm{~mm}^{2}$. Determine the strain energy stored in the bar as shown in Fig 83. | 10 | CO8 | L3 |
| 32 | A bar of length 400 mm and diameter 50 mm is subjected to a tensile load of 20 KN . Find the stress, elongation and strain energy produced if the load is applied gradually. What would be the instantaneous stress and elongation if the same load is applied suddenly. Take $\mathrm{E}=200 \mathrm{GPa}$. | 10 | CO8 | L3 |
| 33 | A weight of 2 KN falls 24 mm on to a collar fixed to a steel bar 14 mm in diameter and 5.5 m long. Determine he maximum sress induced in the bar due to impact. $\mathrm{E}=205 \mathrm{GPa}$ | 10 | CO8 | L3 |
| 34 | Calculate the strain energy stored in a bar 2 m long, 50 mm wide and 40 mm thick, when it is subjected to tensile load of 50 KN . Take $\mathrm{E}=$ 200 GPa . | 10 | CO8 | L3 |
| 35 | A Rectangular body 500 mm long, 100 mm wide and 50 mm thick is subjected to a shear stress of 80 MPa . Determine the strain energy stored in the body, $\mathrm{G}=85 \mathrm{GPa}$. | 10 | CO8 | L3 |
| 36 | A stepped steel rod is in two parts, one has diameter 25 mm . The box is lowered at the rate of $1 \mathrm{~m} / \mathrm{sec}$. When the freely hanging rope is 12 m , the machine gets load is brought to a stop instantaneously. Find the maximum stress in the rope and the instantaneous deformation due to jamming. Take $\mathrm{E}=200 \mathrm{GPa}$. | 10 | CO8 | L3 |
| 37 | A cantilever beam is loaded as shown in Fig. Determine the deflection and slope at the free end by using Castigliano's theorem. | 10 | CO8 | L3 |
| 38 | A shaft is in two parts as shown in Fig. Find the total strain energy stored in it Take G $=85 \mathrm{GPa}$. | 10 | CO8 | L3 |
| 39 | A simple beam ACB supporting an uniform load w over the first half of the beam and a couple of moment $\mathrm{M}_{\mathrm{o}}$ at end B is shown in Fig. 91. Determine the strain energy $U$ stored in the beam due to the load $w$ and the couple $\mathrm{M}_{\mathrm{o}}$ acting simultaneously. | 10 | CO8 | L3 |
| 40 | A cantilever beam of uniform cross section carries a point load at the free end. Determine strain energy and deflection at the free end. If $\mathrm{F}=$ $200 \mathrm{kN}, \mathrm{E}=200 \mathrm{GPa}, \mathrm{L}=3 \mathrm{mt}$ and $\mathrm{I}=10^{-4} \mathrm{~m}^{4}$. | 10 | CO8 | L3 |
| 41 | Two bars, each of length and of different materials are each subjected to the same tensile force P . The first bar has a uniform diameter ' D ' and the second bar has a diameter of $\mathrm{D} / 2$ for a length $\mathrm{L} / 4$ and a diameter D for the remaining length. Compare the strain energies of the two bars if, <br> (i) $\mathrm{E} 1 / \mathrm{E} 2=4 / 7$ and <br> (ii) E 1 = E2 | 10 | CO8 | L3 |
| 42 | Define strain energy and Resilience | 10 | CO8 | L2 |

## D3. TEACHING PLAN - 3

Module - 3

| Title: | Shear force and Bending Moment Diagrams Bending Stresses in Beams | Appr Time: | 10 Hrs |
| :---: | :---: | :---: | :---: |
| a | Course Outcomes | CO | Blooms |
| - | At the end of the topic the student should be able to ... |  | Level |
| 1 | Draw shear force diagram \& Bending moment diagram for a Simply supported beam \& cantilever beam with point load, UDL \& UVL | CO9 | L3 |
| 2 | Determine the bending stress distribution of rectangular, symmetrical I \& T section | CO10 | L3 |
|  |  |  |  |
|  |  |  |  |
|  |  |  |  |
| b | Course Schedule | - |  |
| Class No | Portion covered per hour | - |  |
| 41 | Types of beams, Loads and reactions, | C09 | L2 |
| 42 | Relationship between loads, shear forces and bending moments | CO9 | L2 |
| 43 | Shear force bending moments of cantilever beams subjected to concentrated loads, UDL / UVL | CO9 | L3 |
| 44 | Shear force bending pin support subjected to concentrated loads, UDL / UVL | C09 | L3 |
| 45 | Shear force roller supported beam subjected to concentrated loads, UDL / UVL | CO9 | L3 |
| 46 | Pure bending, curvature of beam | CO10 | L2 |
| 47 | Longitudinal strains in beams | CO10 | L2 |
| 48 | Bending stresses in beams with rectangular , flexure formula and bending stress | CO10 | L3 |
| 49 | Bending stresses in beams with circular, flexure formula and bending stress | CO10 | L3 |
| 50 | Bending stresses in beams with ' I ' and ' T ' cross section, flexure formula and bending stress | CO10 | L3 |
|  |  |  |  |
|  |  |  |  |
| c | Application Areas | - | - |
| - | Students should be able employ / apply the Module learnings to ... | - | - |
| 1 | Design of Bridges, | CO9 | L3 |
| 2 | Structural analysis in the field of construction of bridges, etc. | CO10 | L3 |
|  |  |  |  |
|  |  |  |  |
|  |  |  |  |

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| d | Review Questions | - | - |
| :---: | :---: | :---: | :---: |
| - | The attainment of the module learning assessed through following questions | - | - |
| 79 | What is a beam? What do you mean by statically indeterminate beam? | CO9 | L2 |
| 80 | How are beams classified? Give a brief account. | CO9 | L2 |
| 81 | What are the main type of supports? Distinguish between roller and hinged supports. | CO9 | L2 |
| 82 | Define the terms Axial force, Shear force, bending moment and point of contraflexure. | CO9 | L2 |
| 83 | Explain the sign convection for axial force, Shear force and bending moment. | CO9 | L2 |
| 84 | What are sagging and hogging bending moments? | CO9 | L2 |
| 85 | Derive the relation between intensity of loading, shear force and bending moment in a beam. | CO10 | L2 |
| 86 | The bending moment in a beam is maximum or minimum where shear force is zero. Is the converse true? Why? | CO10 | L2 |
| 87 | Indicate the shape of shear force diagram in case of uniformly distributed load and for triangular loads. | CO10 | L2 |
| 88 | For the beam shown in Fig. Draw the SFD and BMD. Determine the maximum BM and point of Contraflexure. | CO9 | L3 |
| 89 | A simply supported beam is shown in Fig.46. Draw SFD \& BMD | CO9 | L3 |
| 90 | Find the reactions at the fixed end draw the shear force diagram and bending moment diagram for the beam loaded as shown in Fig. Locate the point of contraflexure if any. | CO9 | L3 |
| 91 | A cantilever 2 m long is loaded with uniformally distributed load of $10 \mathrm{KN} / \mathrm{m}$ run over a length of 1.5 m from the free end. It also carries a point load of 10 KN at a distance of 0.5 $m$ from the free end. Find the reaction st the fixed end and draw the SFD and BMD. | CO9 | L3 |
| 92 | Derive the relationship between load shear force and bending moment for UDL. | CO9 | L2 |
| 93 | Derive bending equation $\frac{M}{I}=\frac{\sigma}{y}=\frac{E}{R}$. | CO10 | L3 |
| 94 | A simply supported beam of span 5 m has a cross section $150 \mathrm{~mm} \times 250 \mathrm{~mm}$. If the permissible stress is $10 \mathrm{~N} / \mathrm{mm}^{2}$, find (I) maximum intensity of uniformly distributed load it can carry, (ii) maximum concentrated load P applied at 2 m from an end it can carry | CO10 | L3 |
| 95 | A timber cantilever 200 mm wide and 300 mm deep is 3 m long. It is loaded with a UDL of $3 \mathrm{KN} / \mathrm{m}$ over the entire length. A point load of 2.7 KN is placed at the free end of the cantilever. Find the maximum bending stress produced. | CO10 | L3 |
| 96 | A rectangular beam 300 mm deep is simply supported over a span of 4 m . What uniformly distributed load per meter the beam may carry. If the bending stress is not to exceed 120 $\mathrm{N} / \mathrm{mm}^{2}$ take $\mathrm{I}=8 \times 10^{6} \mathrm{~mm}^{4}$. | CO10 | L3 |
| 97 | Prove that in case of a rectangular section of a beam, the maximum shear stress is 1.5 times average shear stress. | CO10 | L3 |
| 98 | A rolled steel joint, simply supported across a span of 4 m and carrying a UDL of 80 $\mathrm{KN} / \mathrm{m}$, has the following dimensions. Overall depth 350 mm , each flange $150 \mathrm{~mm} \times 25$ mm and web $300 \times 12 \mathrm{~mm}$. Determine the magnitude of the bending and shearing stresses at the junction of the web with the top flange at a section 1 m away from the support. | CO10 | L3 |
| e | Experiences | - | - |
| 1 |  | CO10 | L2 |
| 2 |  | CO9 |  |
| 3 |  |  |  |
| 4 |  | CO9 | L2 |
| 5 |  |  |  |

## E3. CIA EXAM - 3

a. Model Question Paper - 3

| Crs |  | 18ME32 | Sem: | III | Marks: | 30 | Time: | 75 m | minutes |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Course: |  | Mechanics of Material |  |  |  |  |  |  |  |  |  |
| - | - | Note: Answer all questions, each carry equal marks. Module : 3 |  |  |  |  |  |  | Marks | CO | Level |
| 1 | a | For the beam shown in Fig. Draw the SFD and BMD. Determine the maximum BM and point of Contraflexure. |  |  |  |  |  |  | 8 | CO9 | L3 |
|  | b | Derive bending equation $\frac{M}{I}=\frac{\sigma}{y}=\frac{E}{R}$ |  |  |  |  |  |  | 7 | CO10 | L2 |
|  |  | OR |  |  |  |  |  |  |  |  |  |
| 2 |  | A cantilever 2 m long is loaded with uniformally distributed load of $10 \mathrm{KN} / \mathrm{m}$ run over a length of 1.5 m from the free end and draw the SFD and BMD. |  |  |  |  |  |  | 8 | CO9 | L3 |
|  | b | Prove that in case of a rectangular section of a beam, the maximum shear stress is 1.5 times average shear stress. |  |  |  |  |  |  | 7 | CO10 | L2 |
|  | a | Derive the relation between intensity of loading, shear force and bending moment in a beam. |  |  |  |  |  |  |  |  |  |
| 3 |  |  |  |  |  |  |  |  | 8 | CO9 | L2 |
|  | b | A simply supported beam of span 5 m has a cross section $150 \mathrm{~mm} \times 250 \mathrm{~mm}$. If the permissible stress is $10 \mathrm{~N} / \mathrm{mm}^{2}$, find (I) maximum intensity of uniformly distributed load it can carry, (ii) maximum concentrated load P applied at 2 m from an end it can carry |  |  |  |  |  |  | 7 | CO10 | L3 |
|  |  | OR |  |  |  |  |  |  |  |  |  |
| 4 | a | Define the terms Axial force, Shear force, bending moment and point of contraflexure. |  |  |  |  |  |  | 7 | CO9 | L2 |
|  | b | A rolled steel joint, simply supported across a span of 4 m and carrying a UDL of 80 $\mathrm{KN} / \mathrm{m}$, has the following dimensions. Overall depth 350 mm , each flange 150 mm x 25 mm and web $300 \times 12 \mathrm{~mm}$. Determine the magnitude of the bending and shearing stresses at the junction of the web with the top flange at a section 1 m away from the support. |  |  |  |  |  |  | 8 | CO10 | L3 |

## b. Assignment - 3

Note: A distinct assignment to be assigned to each student.

| Model Assignment Questions |  |  |  |  |  |  |  |
| :--- | :--- | :--- | :--- | :--- | :--- | :--- | :---: |
| Crs Code: | 18ME32 | Sem: | III | Marks: | 10 | Time: |  |
| Course: | Mechanics of Materials |  | Module $: 3$ | $90-120$ minutes |  |  |  |

Note: Each student to answer 2-3 assignments. Each assignment carries equal mark.






## F. EXAM PREPARATION

## 1. University Model Question Paper



|  | thickness is calculated based on thin cylinder. |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
| 5 | a | Show that a hollow circular shaft whose inner diameter is half the outer diameter has a torsional strength equal to $15 / 16$ of that of a solid shaft of same outside diameter. | 12 | CO5 | L3 |
|  | b | Explain the following theories of failure.(i) Maximum normal stress theory (ii) Maximum shear stress theory. | 8 | CO6 | L2 |
|  |  | OR |  |  |  |
| 6 | a | Derive the relation for a circular shaft when subjected to torsion as given below $\frac{T}{J p}=\frac{\tau}{R}=\frac{G \theta}{l}$ | 12 | CO5 | L3 |
|  | b | What do you mean by theories of failure? What is their importance? | 8 | CO6 | L2 |
| 7 | a | A 2 m long column with hollow circular section of outer diameter 180 mm is subjected to a load of 212.5 KN . Yield stress for the column material is $340 \mathrm{~N} / \mathrm{mm}^{2}$. One end of the column is fixed and the other end is free. Determine the inner diameter, taking $\mathrm{E}=210 \mathrm{GPa}$ and $\mathrm{FOS}=4$ | 10 | CO7 | L3 |
|  | b | Derive the expression for Euler's critical load for a long column with both ends fixed. | 10 | CO8 | L2 |
|  |  | OR |  |  |  |
| 8 | a | A cast iron column 140 mm external diameter and 70 mm internal diameter is 3.5 m long. Calculate the safe axial load the column can carry if both ends are fixed. Take $\sigma_{c}=600 \mathrm{MPa} ; \alpha=\frac{1}{1600}$ and factor of safety $=3$ use Rankine's formula | 10 | CO7 | L3 |
|  | b | A 1.6 m long bar is applied an axial pull such that the maximum stress induced is 140 MPa . The larger and smaller area of cross section are $240 \mathrm{~mm}^{2}$ and $120 \mathrm{~mm}^{2}$. Determine the strain energy stored in the bar as shown in Fig 83. | 10 | CO8 | L3 |
| 9 | a | Derive the relation between intensity of loading, shear force and bending moment in a beam. | 08 | CO9 | L2 |
|  | b | Derive bending equation $\frac{M}{I}=\frac{\sigma}{y}=\frac{E}{R}$. | 12 | CO9 | L2 |
|  |  | OR |  |  |  |
| 10 | a | A cantilever 2 m long is loaded with uniformly distributed load of $10 \mathrm{KN} / \mathrm{m}$ run over a length of 1.5 m from the free end. It also carries a point load of 10 KN at a distance of 0.5 m from the free end. Find the reaction st the fixed end and draw the SFD and BMD. | 10 | CO9 | L3 |
|  | b | A simply supported beam of span 5 m has a cross section $150 \mathrm{~mm} \times 250 \mathrm{~mm}$. If the permissible stress is $10 \mathrm{~N} / \mathrm{mm}^{2}$, find (I) maximum intensity of uniformly distributed load it can carry, (ii) maximum concentrated load P applied at 2 m from an end it can carry | 10 | CO10 | L3 |

## 2. SEE Important Questions



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|  |  | compressive strength of columns. |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  | d | Define the terms, strain energy, resilience and modulus of resilience. | 8 | CO8 | 2015 |
|  | e | State and prove Castigliano's first theorem. | 8 | CO8 | 2016 |
| 3 | a | Define the terms Axial force, Shear force, bending moment and point of contra flexure. | 6 | CO9 | 2016 |
|  | b | Explain the sign convection for axial force, Shear force and bending moment. | 6 | CO9 | 2015 |
|  | c | What are sagging and hogging bending moments? | 4 | CO9 | 2014 |
|  | d | Derive the relation between intensity of loading, shear force and bending moment in a beam. | 6 | CO9 | 2016 |
|  | e | The bending moment in a beam is maximum or minimum where shear force is zero. Is the converse true? Why? | 5 | CO10 | 2017 |

## G. Content to Course Outcomes

## 1. TLPA Parameters

Table 1: TLPA - Example Course

| Mo |
| :--- | :--- | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| dul |
| e- \# | | Course Content or Syllabus <br> (Split module content into 2 parts which have <br> similar concepts) |
| :---: |
| A |


| with varying circular and rectangular cross-sections subjected to normal and temperature loads |  | $\begin{array}{r} \text { - L2 } \\ -\mathrm{L} 3 \\ \hline \end{array}$ |  |  |  | -CIE <br> -Unit Test |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 2 Plane stress and strain, major and minor principle stresses and their planes, maximum shear stress and their planes, Hoop's stress, Longitudinal stress, Radial stress | 5 | $\begin{aligned} & -\mathrm{L} 2 \\ & \hline \text { L3 } \end{aligned}$ | L3 | Compute | - Lecture | - Assignment <br> -CIE <br> -Unit Test |
| 2 Determine principal stress, maximum shear stress and their orientations using analytical method and Mohr' circle method and dimensions of thick and thin cylinder and also stress distribution in thick and thin cylinders | 5 | $\begin{aligned} & -\mathrm{L} 2 \\ & -\mathrm{L} 3 \end{aligned}$ | L3 | Compute | - Lecture | - Assignment <br> -CIE <br> -Unit Test |
| 4 Circular solid and hallow shafts, Torsional moment of resistance, Power transmission of straight and stepped shafts | 5 | $\begin{aligned} & \text { - L2 } \\ & \hline \text { - L3 } \end{aligned}$ | L3 | Compute | - Lecture | - Assignment <br> -CIE <br> -Unit Test |
| 4 Maximum Principal Stress theory, Maximum shear stress theory | 5 | $\begin{aligned} & -\mathrm{L} 1 \\ & -\mathrm{L} 2 \\ & -\mathrm{L} 3 \end{aligned}$ | L3 | Compute | - Lecture | - Assignment -CIE <br> -Unit Test |
| 5 Buckling and stability, Critical load, Columns with pinned ends, Columns with other support conditions and also dimensions of elastic stability of columns using Rankin's and Euler's theory | 5 | $\begin{aligned} & -\mathrm{L} 2 \\ & -\mathrm{L} 3 \end{aligned}$ | L3 | Compute | - Lecture | - Assignment <br> -CIE <br> -Unit Test |
| 5 Castigliano's theorem I and II, Load deformation diagram, Strain energy due to normal stresses, Shear stresses, Modulus of resilience, Strain energy due to bending and torsion | 5 | $\begin{aligned} & -\mathrm{L} 2 \\ & -\mathrm{L} 3 \end{aligned}$ | L3 | Compute | - Lecture | - Assignment <br> -CIE <br> -Unit Test |
| 3 Pure bending, Curvature of a beam, Longitudinal strains in beams, Bending stresses in Beams with rectangular, 'I' and ' $T$ ' cross sections | 5 | $\begin{aligned} & -\mathrm{L} 1 \\ & -\mathrm{L} 2 \\ & -\mathrm{L} 3 \end{aligned}$ | L3 | Compute | - Lecture | - Assignment <br> -CIE <br> -Unit Test |
| 3 Types of beams, loads, supports, relation between load, force and moment. Draw SFD and BMD for different beams including cantilever beams, simply supported beams and overhanging beams subjected to UDL, UVL, Point loads and couples. | 5 | $\begin{aligned} & -\mathrm{L} 1 \\ & -\mathrm{L} 2 \\ & -\mathrm{L} 3 \end{aligned}$ | L3 | Compute | - Lecture | - Assignment <br> -CIE <br> -Unit Test |

## 2. Concepts and Outcomes:

Table 2: Concept to Outcome - Example Course

| $\begin{aligned} & \mathrm{Mo} \\ & \text { dul } \\ & \mathrm{e}-\# \end{aligned}$ | Learning or <br> Outcome from <br> study of the <br> Content or <br> Syllabus | Identified <br> Concepts from Content | Final Concept | Concept Justification <br> (What all Learning <br> Happened from the <br> study of Content / <br> Syllabus. A short word <br> for learning or <br> outcome) | CO Components (1.Action Verb, 2.Knowledge, 3.Condition / Methodology, 4.Benchmark) |  | Course Outcome <br> Student Should be able to ... |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| A | I | J | K | $L$ | M |  | $N$ |
| 1 | $\begin{aligned} & \hline \text { Stress } \\ & \text {-Strain } \end{aligned}$ | -Uni axial Stress <br> -Uni axial Strain | Uni Axial Stress and Strain | Comprehend the Uni axial stress and Strain and their Relations | -Understand <br> -Stresses strains <br> -stress and st relations |  | Understand simple,  <br> thermal stresses $\&$ <br> strains and their <br> relations   |
| 1 | -Deformation in Temperature stress <br> -Elastic Constants | - <br> Deformatio <br> n <br> -Elastic <br> Constants | Deformation | Comprehend the deformation in varying bar section and elastic constants | -Understand -Determine -Deformation |  | Determine the stress, strain and deformation in bars with varying cross section |
| 2 | -Compound stresses, <br> -Mohr's Circle | -Bi axial stress | Bi axial stress | Comprehend the bi axial stress with analytical and graphical method | -Understand <br> -Principal stress and strain <br> -Analytical and Graphical Method |  | Determine principal stresses and planes using analytical and graphical method. |
| 2 | - Thin Cylinder | -Hoops | Hoops and | Comprehend the | -Understand |  | Determine stress |

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|  | -Thick Cylinder | stress <br> -Radial Stress | Radial Stresses | Hoops stress and radial stresses in thick and thin cylinder | -Hoops stress <br> -Radial stresses | distribution in thick and thin cylinder |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 4 | -Torsion | -Torque \& Twist | Torque \& Twist | Understand the torque and twist in solid and hollow shafts | -Understand -torque and twist -shafts | Determine, $r$ torsional <br> strength, rigidity, <br> flexibility $\&$ <br> dimensions of shaft  |
| 4 | -Theories of Failures | -Theories of failure | Theories of failure | Comprehend the basic concept theories of failures | -Understand -Basic concept -theories of failures | Determine dimensions of bars, beams \& rods using Maximum principal \& Maximum Shear stress theory of failure |
| 5 | - Columns | -Crippling load | Crippling load | Comprehend the crippling load of column in different end condition | -Understand -crippling load -end condition | Determine elastic stability of columns using Rankine's \& Euler's theory |
| 5 | -Stain Energy | -Strain energy methods - | Strain energy methods | Comprehend the strain energy of beams, bars and rods | -Understand -strain energy -beams, bars and rods | Determine dimensions of bars, beams \& rods using strain energy methods |
| 3 | -SFD and BMD | -Shear force, \& Bending moment | Shear force, \& Bending moment | Understand the Concept to draw the SFD and BMD | -Understand -Concept <br> -SFD and BMD | Draw shear force diagram \& Bending moment diagram for a Simply supported beam \& cantilever beam with point load, UDL, UVL and Couple |
| 3 | -Stresses in Beams | -Bending stress distribution | Bending stress distribution | Understand the bending stress distribution of different cross section | -Understand -bending stress -stress distribution of different cross section | Determine the bending stress distribution of rectangular symmetrical I \&T section |

